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# EFFECTS OF FUEL VOLATILITY ON DRIVEABILITY OF 1980 MODEL CARS AT LOW AND INTERMEDIATE AMBIENT TEMPERATURE

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# 1980 MODEL CARS AT LOW AND INTERMEDIATE AMBIENT TEMPERATURE (CRC PROJECT No. CM-118-80)

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Prepared by the
1980 Analysis Panel
of the
CRC-Volatility Group

March 1982

Light-Duty Vehicle Fuel, Lubricant, and Equipment Research Committee

of the

Coordinating Research Council, Inc.

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# I. OBJECTIVE

In previous CRC studies, equations were developed to predict vehicle driveability at intermediate ambient temperatures (nominally 40-70°F) as a function of gasoline distillation properties. Such equations are useful for predicting the driveability performance of commercial gasolines. The purposes of the present study were: 1) to supplement those previous studies at intermediate temperatures with data for 1980 model year vehicles with and without closed-loop fuel control systems; and 2) to determine the relationship between fuel volatility and vehicle driveability at low ambient temperatures (<20°F), thus expanding the applicability of equations for predicting driveability performance.

# II. SUMMARY AND CONCLUSIONS

Cold start and warmup driveability was measured at low (-20 to 28°F) and intermediate (40 to 69°F) ambient temperatures using 1980 model cars (10 with closed-loop and 6 with open-loop fuel systems). Nine gasolines which varied independently in distillation temperatures at 10, 50, and 90% evaporated ( $T_{10}$ ,  $T_{50}$ , and  $T_{90}$ ) were used. Results are described in terms of total weighted demerits (TWD), which were adjusted to reduce the influence of variations among raters. Averages were computed across cars and/or across fuels, as appropriate.

- Average TWD (132) at low ambient temperatures were not significantly different from average TWD (125) at intermediate ambient temperatures. This apparently resulted from the fact that for some cars and fuels, TWD increased as temperature decreased, and for other cars and fuels, TWD decreased as temperature decreased.
- At low temperatures, average TWD with the lowest volatility fuel (Fuel 1) were about double those with the intermediate volatility fuel (Fuel 2), while average TWD with the highest volatility fuel (Fuel 3) were about 30% lower than those with Fuel 2. At intermediate temperatures, effects were similar, but of smaller magnitude.
- At intermediate embient temperatures, the effect of  $T_{10}$ ,  $T_{50}$ , and  $T_{90}$  on driveability was similar to that found in previous CRC intermediate temperature studies. At low embient temperatures,

however, the importance of  $T_{10}$  increased and the importance of  $T_{90}$  decreased relative to that at intermediate temperatures. The equation developed to predict driveability as a function of  $T_{10}$ ,  $T_{50}$ ,  $T_{90}$ , and  $T_{80}$  (ambient temperature in the range of -20 to 69°F) is given below:

TWD\* = -371.7 + 0.0251\*Te + (2.0582 - 0.027171\*Te)\*T10

- + (0.9925 + 0.002245 Ta) Tso
- + (0.1576 + 0.007379\*Ta)\*Tyo

The following table, which has been derived from the above equation, illustrates the effect of ambient temperature on the relative importance of  $T_{10}$ ,  $T_{50}$ , and  $T_{90}$ .

# Normalized Regression Coefficients

Ta, °F	T <sub>10</sub>	Ts o	Too
	<del></del>		
-20	0.73	0.27	0.00
0	0.64	0.31	0.05
20	0.53	0.36	8.11
40	0.39	0.43	0.18
60	0.20	0.52	0.28

The normalized regression coefficients at 40 and 60°F were similar to those previously obtained in a study with 1973 model cars at 40-70°F.

• With the above 8-coefficient equation, the differences between actual and predicted TWD were greatest with the fuels which had distillation curves roughly parallel to those of commercial gasolines. For example, the equation underestimates average TWD with Fuel 1 by about 15 and overestimates average TWD with Fuel 2 by about 20. To improve prediction of TWD, two nonlinear terms (T10°T50 and T50²) were added to the model:

TWD\* = 1894.6 + 0.8759 Ta - (0.0363 + 0.027829 Ta) T<sub>10</sub> + (-19.5514 + 0.002388 Ta + 0.010321 T<sub>10</sub>) T<sub>50</sub> + 0.044768 T<sub>50</sub><sup>2</sup> + (0.2949 + 0.007320 Ta) T<sub>90</sub>

With this 10-coefficient equation, TWD for Fuel 1 were underestimated by only 7 and for Fuel 2 were underestimated by only 3. Because of the nonlinear terms in this equation, it was not possible to derive normalized regression coefficients such as those shown above for the 8-coefficient equation.

\*Notes regarding both regression equations: not all coefficients were significant at the 90% confidence level; indices of determination  $(r^2)$  were low due to large car-to-car differences.

- As in past programs, there were large differences among cars in driveability level and the effects of fuel volatility on driveability. However, there were no significant differences between matched open-loop and closed-loop models. The similarity in driveability between open-loop and closed-loop models apparently results from the fact that closed-loop systems are not operative during warmup.
- TWD levels for these 1980 model cars were greater than those of 1977 model cars, less than those of 1973 model cars; and similar to those of 1975 model cars which have been tested by CRC. The effect of fuel volatility on driveability of these 1980 model cars was similar to that of the 1973 models.
- Classification of demerits by malfunction and test cycle showed that hesitation, stumble, and stalls during accelerations accounted for about 80% of TWD and that most TWD occur during the first 3 cycles of the 6-cycle CRC procedure.
- Of several alternative demerit counting systems which were evaluated, only one (demerits weighted according to a scale developed in a study of customer response to driveability) gave noticeably different regression coefficients than those shown above. Further evaluation of this scale in future programs may be warranted.

# III. INTRODUCTION

Based on a CRC study<sup>1</sup> with 1973 model year cars at intermediate ambient temperatures (40-70°F), the following equation was developed to predict cold start and warmup driveability as a function of gasoline distillation properties:

$$TWD = -285.7 + 0.6166 \cdot T_{10} + 0.8527 \cdot T_{50} + 0.4706 \cdot T_{90}$$
 (1)

where  $T_{10}$ ,  $T_{50}$ , and  $T_{90}$  are the temperatures (°F) at 10, 50, and 90 percent evaporated, respectively. This equation indicates that TWD increase (driveability deteriorates) as  $T_{10}$ ,  $T_{50}$ , and  $T_{90}$  increase, and that the effect of  $T_{50}$  is greater than that of either  $T_{10}$  or  $T_{90}$ . In commercial use, the above equation is often approximated by the following relationship:

TWD 
$$\alpha$$
 0.5°T<sub>10</sub> + T<sub>80</sub> + 0.5°T<sub>90</sub> (2)

A similar CRC program was conducted with 1975 model cars2, but compatibility problems between some of the fuels and fuel systems

prevented the establishment of a new equation. Instead, the data for the fuels giving no compatibility problems were analyzed using the equation for 1973 cars. The equation developed for 1973 cars appeared to also be suitable for 1975 cars.

These preceding relationships have been useful for the prediction of the driveability performance of gasolines at intermediate ambient temperatures. However, for temperatures below 40°F, no published data were available to establish such an equation, and it could not be assumed that the equation developed for intermediate temperatures would be applicable to low temperatures. Consequently, this study was conducted to determine the influence of gasoline volatility on vehicle driveability at low ambient temperatures (-20 to 28°F).

For this study, 1980 model year cars were chosen because the 1980 models sold in California were equipped with closed-loop fuel control systems, while those sold in other states generally had open-loop fuel control systems. Thus, differences between closed-loop and open-loop cars, in driveability level and the effect of gasoline volatility on driveability, could be evaluated.

There also were several reasons for conducting driveability tests at intermediate temperatures in conjunction with those at low temperature:

1) to determine how driveability of 1980 model cars (with and without closed-loop systems) compared to that of previous model year cars studied at intermediate ambient temperatures; 2) to determine if the effect of fuel volatility on driveability was different for 1980 than for previous model years; and 3) to allow direct comparison between low and intermediate temperatures, of driveability level and the effect of fuel volatility on driveability for the same cars operated at the same test location. This report describes the program which was conducted to meet these goals, and the results which were obtained.

# IV. EXPERIMENTAL CONDITIONS AND DATA ANALYSIS TECHNIQUES

A cold-start and warmup driveability program was conducted in Brainerd, Minnesota in two phases: one at low temperature and one at intermediate temperature. The low temperature (nominally <20°F) tests were conducted from January 24 through February 24, 1980 and the intermediate temperature (nominally 40-60°F) tests were conducted from April 15 through May 14, 1980. In each test phase, trained raters evaluated the driveability of sixteen 1980 model cars using nine fuels, each with different volatility. Appendix A lists participants in the test program and program planning and data analysis panel members. The program proposal approved by the CRC Volatility Group is shown in Appendix B.

# A. TEST SITE

Brainerd International Raceway was rented by CRC and used as the test site for the program. A schematic of the test site is shown in Figure 1. The facilities used by CRC included a fuel storage shed, equipment storage room, office space, overnight parking for the test rars, and the track. The driveability tests were conducted only on the front section of the track but cars were preconditioned for the next day's testing by driving them two laps around the entire three-mile track:

This location was chosen because: (1) weather records indicated that ambient temperature would be suitable and precipitation would be minimal, and (2) a suitable test site was available. As will be discussed in the next section, the weather was not as favorable as had been expected.

# B. AMBIENT CONDITIONS

Y.

The distribution of ambient temperatures during each of the test runs is shown in Figure 2. The program proposal listed 0-20°F as target temperatures for the low temperature phase. However, at the time that tests were to start, ambient temperatures rarely exceeded 0°F, so the minimum test temperature was lowered to -20°F. Near the end of the low temperature phase, temperatures were often higher than 20°F. There were three days in which testing was cancelled due to high ambient temperature, and three additional days in which a portion of the tests were run at ambient temperatures above 20°F (up to 28°F).

In addition, the condition of the track was not ideal at the beginning of the low temperature phase. Shortly before the program began, several inches of snow fell in the Brainerd area. The snow partially melted then refroze, leaving an ice coating on the test track. The contractor hired for snow removal was unable to clear the ice, so critical areas were cleared manually, as is shown in Figure 3. Efforts to clear the track were only partially successful, but to minimize delays, testing was begun before the track was cleared. Several days throughout the program, testing had to be delayed a few hours until new snow could be cleared from the track.

Ambient conditions were more favorable during the intermediate temperature phase of the program. Tests were never cancelled due to rain or due to undesired ambient temperatures. There were, however, a total of 29 runs at ambient temperatures above the target of 60°F. The maximum temperature at which tests were run was 69°F.

# C. CARS

Table 1 describes the 16 test cars used in this study. Eight of the cars were General Motors products, 5 were from Ford Motor Co., and 3 from Chrysler Corporation. This distribution roughly approximates the relative U.S. market share of these 3 manufacturers. A variety of model and engine sizes were selected. It was, planned to test only vehicles with automatic transmissions; however, for one of the selected models, the only available car was one with a manual transmission.

Ten of the cars had closed-loop fuel systems\* and 6 had open-loop systems. In general the closed-loop cars were certified to meet California exhaust emissions standards and the open-loop cars were certified to meet 49-state exhaust emissions standards. One closed-loop vehicle was certified to meet emissions standards in all 50 states 3ix of the closed-loop cars were matched by model and engine to the 6 n-loop cars.

Throughout the report the cars will be designated by the number through 16. Table 2 identifies each of the car numbers according fuel system type and the CRC identifying code.

The cars were leased and delivered to Brainerd several days prior to the start of the program to provide time for preparation by a local dealership. Car preparation by the dealer included:

- a check of spark timing, fluid levels, tire pressure, and emission control systems
- installation of an auxiliary fuel line mounted to the front bumper
- installation of a basket for securing a test fuel can to the front bumper (the basket is shown in Figure 4)

Prior to the start of testing, CRC participants operated the cars on the test track to establish the intake manifold vacuums to be used for the light throttle and detent accelerations (see Appendix B).

Between the low and intermediate test phases, the cars were stored in buildings at the track site.

\*Computer-controlled fuel metering based on a signal from an oxygen sensor in the exhaust system.

### D. FUELS

The design set of nine test gasolines used in this program was similar to the test gasolines used in past CRC programs of this type<sup>1</sup>,<sup>2</sup>. They varied independently in the distillation temperatures at which 10, 50, and 90% of the sample had evaporated. A detailed description of the fuels is given in Table 3, and a graphic representation of the fuel design is shown in Figure 5. The approximate temperature range of each fuel design variable was:

<u>Design Variable</u>	Temperature Range, °F
10% Evap. Temperature (T10)	100 - 140
50% Evap. Temperature (Tso)	185 - 245
90% Evap. Temperature (Too)	295 - 360

The volatility ranges were selected to approximate the range for commercial gasolines.

All test fuel for the program was blended before the low temperature phase, but only half of the total quantity of each fuel was shipped to Brainerd for the low temperature phase. The remaining fuel was stored by the blender until the beginning of the second test phase. The fuels were shipped and stored on site in 30-gallon drums. Fuel was dispensed from the drums into 2-gallon "safety" cans. The cans were installed on the front bumper of the vehicle as shown in Figure 4. Fuel was supplied to the engine via an auxiliary fuel line from the safety can to the car's fuel pump inlet.

# E. TEST DESIGN

The actual test design, which is shown in Table 4, differed somewhat from that which was planned (see Appendix B). There were 8 raters (A, B, C, D, E, G, H, I) instead of 6 as in the original design. Three raters participated in only the low temperature test phase, four participated in only the intermediate temperature phase, and one rater participated in both test phases. For the purposes of rater assignment, each of the two test phases was divided into two test blocks of roughly two weeks each. There were two raters on site at the same time; both rated 8 cars each day.

# F. DRIVEABILITY PROCEDURE AND RATING SYSTEM

In both phases, raters evaluated car driveability using the test procedures described in Appendix B. The driving procedure consisted of

a cold start followed by a specific series of acceleration, cruise, deceleration and idle maneuvers. The raters evaluated driveability problems such as hesitiation, stumble, surge, and idle roughness using a subjective scale of "none, trace, moderate, or heavy." Starting times were measured and the number of stalls were recorded. Based on the malfunctions observed, demerits were assigned, weighted, and totaled as described in Appendix B to provide total weighted demerits (TWD). As driveability deteriorates TWD increase. Examples of weighted demerit assignments are: 32 for an acceleration stall, 16 for a moderate hesitation, and I for trace idle roughness.

There were two modifications to the driveability procedure for this test program. The first involved starting. Early in the low temperature phase, there were problems with starting some cars due to insufficient battery charge. To avoid this problem during the remainder of the low temperature phase, an auxiliary battery was used to assist in cold cranking (see Figure 4). The second modification was for testing the car with the manual transmission. The driving and shifting procedures for this car are described in Table 5.

# F. DATA ANALYSIS

A brief discussion of the methods used for computing means, analysis of variance, and regression analysis is given below. The technique for adjusting the data to reduce rater effects is discussed later.

# 1. Means

A few comments about the method used for computing means are in order. First, replicate results within each data cell defined by test phase, car, fuel, and rater block (2°16°9°2 = 576 cells) were averaged to provide a balanced data set. Means by car, fuel, and test phase were computed from this balanced data set. For analysis of variance and regression, the entire unbalanced data set was used.

# 2. Analysis of Variance

The significance of car, fuel, and test phase effects was evaluated using analysis of variance with the following model:

TWD = 
$$u + C_i + F_j + P_k + CF_{ij} + CP_{ik} + FP_{jk} + CFP_{ijk} + \varepsilon_{(ijk)} z$$
 (3)

 $\mu$  = the grand mean of TWD

 $C_i$  = the effect of the i\th car (i = 1-16)

 $F_j^{\uparrow}$  = the effect of the j\th fuel (j = 1-9)  $P_k^{\downarrow}$  = the effect of the k\th test phase (k = low, intermediate)

CF<sub>ij</sub> = the effect of the interaction between the i\th car and
the j\th fuel

FP<sub>jk</sub> = the effect of the interaction between the j\th fuel and the k\th test phase

 $\epsilon_{(ijk)}$ ? = random error resulting from the l\th replicate with the i\th car, j\th fuel, and k\th test phase

For this model  $C_1$  is a random variable because the individual cars were obtained at random from the population of each selected make and model. Previous driveability programs 1,2 showed that driveability varies among cars of the same make and model. Fuel and test phase were considered fixed variables in this model.

To determine the significance of all of the terms in the model, F tests were performed. Based on the above model with car as a random variable, the following were used to compute F values for each of the effects:

Effect	F Calculation			
Car	MS(C)/MS(E)			
Fuel	MS(F)/MS(CF)			
Test Phase	MS(P)/MS(CP)			
Car by Fuel	MS(CF)/MS(E)			
Car by Test Phase	MS(CP)/MS(E)			
Fuel by Test Phase	MS(FP)/MS(CFP)			
Car by Fuel by Test Phase	MS(CFP)/MS(E)			

where MS is the mean squares (sum of squares divided by degrees of freedom) for each term. Confidence levels were computed for each F value; levels greater than 90% were considered significant.

To determine the influence of open-loop versus closed-loop fuel systems on driveability, a different model was required:

TWD = 
$$\mu$$
 +  $S_i$  +  $C_{(i)j}$  +  $F_k$  +  $P_l$  +  $SF_{ik}$  +  $SP_{il}$  +  $FP_{kl}$  +
$$FC_{(i)jk} + CP_{(i)jl} + SFP_{ikl} + CFP_{(i)jkl} + \epsilon_{(ijkl)m}$$
(4)

# where:

μ = the grand mean of TWD

 $C_{(i)j}$  = the effect of the j\th car within the group defined by the i\th fuel system

 $F_k$  = the effect of the k\th fuel

 $P_7$  = the effect of the l\th test phase

SFik = the effect of the interaction between the inth system and the kith fuel

 $SP_{ij}$  = the effect of the interaction between the inth system and

the 2xth test phase

 $FP_{kl}$  = the effect of the interaction between the k\th fuel and the l\th test phase

 $FC_{(i)jk}$  = the effect of the interaction between the k\th fuel and the j\th car within the i\th system

CP(i) j? = the effect of the interaction between the j\th car
within the i\th system and the !\th test phase

SFP<sub>ik?</sub> \* the effect of the interaction between the i\th system, the k\th fuel, and the ?\th test phase

CFP(i)jk? = the effect of the interaction between the j\th car
within the i\th system, the k\th fuel, and the i\th
test phase

c(ijkl)m = random error resulting from the m\th replicate
with the i\th system, the j\th car, the k\th fuel,
and the l\th test phase

Again  $C_{(i)j}$  is considered a random variable and all others are considered fixed.  $C_{(i)j}$  is also a nested variable because individual cars can belong to only one of the car groups, i.e. it is either a closed-loop or an open-loop car. F values were computed as shown below:

Effect	F Calculation			
System	MS(S)/MS(C)			
Car	MS(C)/MS(E)			
Fuel	MS(F)/MS(FC)			
Test Phase	MS(P)/MS(CP)			
System by Fuel	MS(SF)/MS(FC)			
System by Test Phase	MS(SP)/MS(CP)			
Fuel by Test Phase	MS(FP)/MS(CFP)			
Fuel by Car	MS(FC)/MS(E)			
Car by Test Phase	MS(CP)/MS(E)			
System by Fuel by Test Phase	MS(SFP)/MS(CFP)			
Car by Fuel by Test Phase	MS(CFP)/MS(E)			

# 3. Regression Analysis

Regression analysis was used to develop equations for predicting driveability as a function of fuel properties and ambient temperature. A standard, least-squares multiple regression technique was used. The models for the analysis, which are described later, did not include terms to account for differences among cars. Thus, these car-to-car variations became part of the error term in the models, and result in relatively high coefficients of variation (standard error expressed as a percent of the mean) and relatively low indices of determination (r²). The significance of all terms in the regression models was evaluated using t-tests.

# V. RESULTS AND DISCUSSION

Detailed results of the driveability tests at low and intermediate temperature are given in Appendix Table C-1. Average TWD (raw) are summarized in Appendix Table C-2. As in past CRC programs, the first step in the data analysis was an attempt to reduce the influence of differences among raters.

# A. RATER CORRECTIONS AND TEST PRECISION STATEMENTS

In previous CRC driveability programs 1,2,3,4 rater corrections were developed to reduce the data bias caused by differences among raters. In the analysis of results from this study, there were similar efforts to identify and remove these rater effects. The method used in this study was derived based on the test design and, consequently, was somewhat different from that used in previous CRC studies. A brief description of the technique is given here. Details are in Appendix D.

The form of the equation for rater correction was:

Adjusted TWD = Raw TWD + C

(5)

As is described in Appendix D, there were four steps in the process of determing the correction constants (C's) for each rater. The first step was to determine, by regression analysis, the average TWD difference between raters on site together. In the low temperature phase, average TWD differences were calculated for Rater A versus Rater B and for Rater C versus Rater D. In the intermediate temperature testing, differences for Rater D versus E, for Rater D versus G, and for Rater H versus Rater I were computed.

Second, within each test phase, an average difference between raters not on site together was determined. This involved finding the Rater A versus Rater D difference in the low temperature phase and finding the Rater D versus Rater H difference for the intermediate temperature phase.

While not all of these rater offsets were statistically significant, they were the best available estimate of the correct offset and therefore were all used in the calculation of rater correction factors.

The third step in developing the correction factors was to calculate the adjustment necessary to equalize the individual rater averages. Because only Rater D participated in both the low and intermediate temperature phases, it was decided that all other raters should be adjusted to make their average TWD equal to the Rater D average.

Applying these corrections to the data increased the overall average TWD by 1.3 TWD, indicating that rater D was about 1.3 TWD more severe than the average rater. Therefore, the fourth step was to subtract 1.3 TWD from all observed ratings by reducing each rater correction by 1.3 TWD. The final rater corrections are:

Rater	C
	,
λ	17.0
В	-15.7
C	-31.3
D	-1.3
E	-20.0
G	16.2
н	34.4
I	-1.2

Adjusted TWD data for each run are shown in Appendix Table C-1. Average results are summarized in Table 6.

Application of these rater corrections reduced the reproducibility standard error\* from 55.5 to 51.5 TWD for the low temperature phase and from 40.9 to 35.3 TWD for the intermediate temperature phase. Test repeatability was 49.1 and 33.9 TWD in the low and intermediate temperature phases, respectively. (An explanation of these calculations is included in Appendix D).

A few additional comments on the rater correction factors are in order. The factors are based on the assumption that the severity of Rater D was the same in both phases of the program. This assumption seems valid, but it cannot be proven (nor disproven) because Rater D did not conduct any tests under identical conditions in both phases.

Because the correction for some raters was negative, the corrected TWD were sometimes negative. Even though TWD less than zero are technically impossible, they were retained and used in all remaining analyses to insure that the effects of fuel volatility on driveability were properly assessed. As will be shown later, these rater adjustments had little effect on the equations used to describe the effect of fuel volatility on driveability.

# B. EFFECTS OF TEMPERATURE (TEST PHASE), FUELS, AND CARS

The results in Table 6 show differences in driveability among the cars and fuels as well as differences between low and intermediate temperature test phases. To determine whether these differences were

<sup>\*</sup> This includes the random variable components: repeatability, day-to-day variability, and rater-to-rater variability.

significant, an analysis of variance was conducted using the model described earlier. Table 7, which summarizes results of the analysis of variance, indicates that there were significant differences in driveability among the cars and fuels selected for this study, and that the interaction between car and fuel was significant. Although differences between the test phases were not significant as a main effect, there were significant interactions between car and test phase and between fuel and test phase. Discussion of each of these effects is given below.

# 1. Temperature (Test Phase) Effects

Average TWD (across cars and fuels) for the low temperature phase were 132.3, about 8 demerits greater than the 124.5 average for the intermediate temperature phase. Considering the large effect of temperature on fuel vaporization, a larger difference between low and intermediate temperature driveability might have been expected.

However, it is important to note that there were highly significant interactions between fuel and test phase and between car and test phase, indicating that the effect of temperature varied among the fuels and cars in this study. To more adequately understand the observed temperature effect, it is important to examine results for individual fuels and cars.

# 2. Fuel Effects

Figure 6 shows TWD averaged across cars for each fuel and test phase. The fuels are identified by number and by relative  $T_{10}$ ,  $T_{50}$ , and  $T_{90}$  values. Results averaged across fuels for each test phase are also shown.

The expected volatility effect is illustrated by the three fuels with nominally parallel distillation curves, Fuels 1, 2, and 3. At low temperature, for example, TWD with the low volatility fuel (Fuel 1) were nearly double those with the intermediate fuel (Fuel 2), while TWD with the high volatility fuel (Fuel 3) were about 30% lower than those with Fuel 2. At intermediate temperature the same effect was apparent, but the magnitude of the differences between Fuels 1, 2, and 3 was smaller than that at low temperature.

Figure 6 indicates that the effects of temperature on driveability were different for the different fuels. For most fuels, driveability was worse (TWD were higher) at low temperature than at intermediate temperature. However, there were 3 fuels, Fuels 3, 4, and 7, for which driveability was better at low than at intermediate temperature. The only common characteristic among these three fuels is their low  $T_{10}$  values.

To illustrate the relative importance of  $T_{10}$  on low and intermediate temperature driveability, Figure 7, a plot of TWD versus  $T_{10}$ , was constructed for fuels with common values of  $T_{50}$  and  $T_{90}$ . For each test phase there are four pairs of data points representing all four possible combinations of high and low  $T_{50}$  and  $T_{90}$ . Each data pair shows increased TWD with increasing  $T_{10}$ , but the effect is noticeably more pronounced at low temperature, as indicated by the steeper slopes of the lines for low temperature in Figure 7. In a subsequent section this effect will be quantified using regression analysis.

# 3. Individual Car Effects

Results averaged across fuels for each car and test phase are shown in Figure 8. Nine cars had higher TWD at low temperature than at intermediate temperature, but the remaining 7 had higher TWD at intermediate temperature.

The large differences among cars in the effects of ambient temperature on driveability probably result from differences in engine calibration. Engine design factors which affect driveability include air-fuel ratio and exhaust gas recirculation (EGR). Calibrations for air-fuel ratio and EGR vary with ambient temperature; these variations could have a significant influence on the effect of temperature on driveability. Thus, the functional relationship between driveability and temperature depends on the air-fuel ratio and EGR calibration of a particular car.

The cars are grouped in Figure 8 according to fuel system (open loop and closed loop). Although there were large differences among cars in driveability level and the effect of ambient temperature on driveability, there were no apparent differences between open-loop and closed-loop cars. Effects of open-loop versus closed-loop fuel systems will be investigated in more detail in the following section.

# 4. Effects of Vehicle Fuel System (Open vs. Closed Loop)

Since differences among cars were so large, analysis of effects of fuel system were conducted using only the closed-loop cars for which there were matching open-loop models (see Table 2). For this set of matched cars, TWD were averaged by fuel system for each fuel and test phase, and are shown in Figure 9. At intermediate temperature, average TWD for the closed-loop cars were lower than those for the open-loop cars, but at low temperature, average TWD for the closed-loop cars were higher than those for the open-loop cars. The effect of changes in fuel volatility appeared to be similar for open-loop and closed-loop cars.

To determine the significance of the differences shown in Figure 9, analysis of variance was conducted using the model described earlier, and the results are summarized in Table 8. According to the results of

this analysis, driveability of open-loop cars was not significantly different from that of closed-loop cars and there were no significant interactions involving fuel system. This lack of significant effect of fuel system probably results from the fact that closed-loop controls are generally not operative during warmup.

# 5. Comparison to Previous Model Year Vehicles

To show trends in driveability level and the response of driveability to changes in fuel volatility, the results of this study with 1980 cars were compared to similar data for 1973 cars<sup>1</sup>, 1975 cars<sup>2</sup>, and 1977 cars<sup>3</sup>. For the 1980 cars, only the intermediate temperature data were used because the previous studies were all conducted at intermediate temperature.

Figure 10 shows TWD plotted versus fuel volatility (expressed in terms of  $0.5 \cdot T_{10} + T_{50} + 0.5 \cdot T_{90}$ ) for each of the 4 studies. TWD levels for these 1980 cars were greater than those of the 1977 cars, less than those of the 1973 cars, and similar to those of the 1975 cars which were tested. The slope of the line through the data points in Figure 10 represents the effect of fuel volatility on driveability; this slope was similar for 1980 and 1973 model cars. Relationships between fuel volatility and driveability will be explored in more detail in the following section.

# C. EQUATIONS FOR PREDICTING DRIVEABILITY AS A FUNCTION OF FUEL VOLATILITY

The fuels used in this study were selected so that regression analysis could be used to develop equations for predicting driveability as a function of the fuel distillation temperatures,  $T_{10}$ ,  $T_{50}$ , and  $T_{90}$ . Various regressions were conducted; they are summarized in Tables 9 and 10, and will be discussed below.

# 1. Separate Equations for Each Test Phase

In previous CRC studies, equations for predicting gasoline driveability performance were based on intermediate temperature data only. To permit comparison between this and past programs, an equation was developed for just the intermediate temperature phase of this study:

# Intermediate Temperature

THD =  $-386.5 + 0.6664 \cdot T_{10} + 1.1285 \cdot T_{50} + 0.5770 \cdot T_{90}$ 

Equation 6 is similar, but not identical, to that obtained for 1973 model cars (equation 1). The magnitude of the slope coefficients and the absolute value of the intercept were all somewhat greater in equation 6 than in equation 1. However, the relative sizes of the slope coefficients were very similar, as is shown by the normalized forms of the equations:

# Intermediate Temperature

TWD (1973 models) = 
$$-285.7 + 1.9399 \cdot (0.32 \cdot T_{10} + 0.44 \cdot T_{50} + 0.24 \cdot T_{90})$$
 (1a)

TWD (1980 models) = 
$$-386.5 + 2.3719 \cdot (0.28 \cdot T_{10} + 0.48 \cdot T_{50} + 0.24 \cdot T_{90})$$
 (6m)

These equations indicate that the relative importance of  $T_{10}$ ,  $T_{50}$ , and  $T_{90}$  on driveability performance at intermediate temperature was quite similar for 1973 and 1980 model cars.

An equation was also developed for the low temperature phase of this program. The convention 1 and normalized forms of the equation are shown below:

#### Low Temperature

$$TWD = -351.3 + 1.9887 \cdot T_{10} + 0.9631 \cdot T_{50} + 0.1364 \cdot T_{90}$$

$$TWD = -351.1 + 3.0882 \cdot (0.64 \cdot T_{10} + 0.31 \cdot T_{50} + 0.04 \cdot T_{90})$$

$$(7a)$$

Comparing equation 7a to equation 6a shows that the  $T_{10}$  coefficient is much greater at low than at intermediate temperature, indicating that the effect of  $T_{10}$  on driveability was greater at low than at intermediate temperature. Conversely, the effect of  $T_{90}$  on driveability was much smaller at low than at intermediate temperature. The effect of  $T_{90}$  on driveability was also smaller at low than at intermediate temperature.

These separate regression analyses for the low and intermediate temperature data indicate that as ambient temperature decreases, the front-end of the distillation curve becomes more important, while the tail-end of the distillation curve becomes less important. If one theorizes that, at a given ambient temperature, driveability is primarily influenced by the amount of vapor formed in the intake manifold, these results can be explained. Suppose that, at a given driving condition at intermediate temperature, about one-half of the gasoline is vaporized. Thus, differences in the volatility properties of the mid-range of the gasoline are probably most important. For the same driving condition at low temperature, less than one-half of the gasoline would be expected to vaporize. Consequently, the front-end volatility characteristics of the fuel would be more important than the mid-range characteristics. Differences in the tail-end of the fuel distillation curve would be of little importance at low temperature because, for the range of fuel volatilities tested, the amount of fuel

vaporized probably never approaches 90% during this critical driving condition.

# 2. Combined Equation for Both Test Phases

It is apparent that equation 6 could be used to describe the driveability performance of these cars at temperatures near 50°F, for example, and equation 7 could be used at temperatures near 0°F. For temperatures between these, such as 30°F, neither equation applies. To allow prediction of driveability performance of gasoline over a range of low and intermediate temperatures, data from both test phases were used to construct a new regression model which included ambient temperature as a variable.

Regression analysis was performed using all of the data from both test phases, and the ambient temperature during each run, Ta (°F), was included in the model as a main effect and as an interaction with  $T_{10}$ ,  $T_{50}$ , and  $T_{90}$  (see Model B in Table 9). The resulting equation is shown below:

TWD = 
$$-371.7 + 0.0251 \cdot Ta + (2.0582 - 0.027171 \cdot Ta) \cdot T_{10} + (0.9925 + 0.002245 \cdot Ta) \cdot T_{50} + (0.1576 + 0.007379 \cdot Ta) \cdot T_{90}$$
(8)

Based on this regression equation, several observations can be made. The effect of ambient temperature on driveability was quite small and depended on fuel properties. Calculated TWD for Fuels 1, 2, and 3 at various temperatures are shown below:

	Calculated TWD (Equation 8)					
IA.ºI	Fuel 1	Fuel 2	Fuel 3			
-20	204.6	132.6	65.6			
0	196.4	129.9	66.2			
20	188.1	127.3	66.8			
40	179.9	124.6	67.3			
60	171.6	121.9	67.8			

Increasing ambient temperature from -20 to 60°F decreased calculated TMD by 33 with Fuel 1, decreased calculated TMD by 11 with Fuel 2, and increased calculated TMD by 2 with Fuel 3.

To demonstrate the effect of ambient temperature on the relative importance of  $T_{10}$ ,  $T_{50}$ , and  $T_{90}$ , normalized regression coefficients were

computed (based on equation 8) for various temperatures; results are shown in the following table:

MODEL: TWD =  $b_0 + (b_1 \cdot T_{10} + b_2 \cdot T_{50} + b_3 \cdot T_{90}) \cdot c$ 

Ta,°F	рo	<b>b</b> <sub>1</sub>	bz	Ъэ	C
		<del></del>			
-20	-372.2	0.73	0.27	0.00	3.559
0	-371.7	0.64	0.31	0.05	3.208
20	-371.2	0.53	0.36	0.11	2.857
40	-370.7	0.39	0.43	0.18	2.506
60	-370.1	0.20	0.52	0.28	2.155

These coefficients show that driveability deteriorated with increasing  $T_{10}$ , but the effect became smaller as ambient temperature increased. Driveability also deteriorated with increasing  $T_{50}$  and  $T_{90}$ , and the effect became larger as ambient temperature increased.

Equation 8 was developed based on data obtained at ambient temperatures ranging from -20 to 69°F. This equation should not be used to predict the driveability performance of these cars outside of this temperature range. Based on this study alone, no judgment can be made on the applicability of this equation to cars other than those tested. However, the similarity of the normalized coefficients at 40 and 60°F to those developed for 1973 models at 40-70°F (equation la) is encouraging. Differences among cars are explored in the next section.

# 3. Equations for Individual Cars and Car Groups

The regression equations which have been presented above were obtained using the combined data for all cars. To determine differences in effects between cars and car groups, separate regression equations were developed for each car and for the open and closed-loop car groups. The results of these regressions are summarized in Table 10.

In general there were substantial differences in regression equations among the individual cars, but there were some similarities. Coefficients for  $T_{10}$  and  $T_{50}$  varied among the cars but were always positive. The coefficients for  $T_{10}$  Ta were negative (indicating increased importance of  $T_{10}$  at low temperature) for all cars. For other terms, the coefficients were positive for some cars and negative for others.

The regression equation for open-loop cars was not significantly different from that for the matched closed-loop cars. This supports the previous analysis which showed no significant differences between open-loop and closed-loop cars in the effect of fuel volatility on driveability.

# 4. Equations for Fuel Volatility Expressed as Percent Evaporated

Some refiners control fuel volatility based on the percent of gasoline evaporated at a given temperature instead of the temperature for a given percent evaporated. To meet the needs of these refiners, several regressions were run with fuel volatility expressed as functions of the percent evaporated at 115, 135, 158, 215, 230, 300, and 330°F. Three such regression were run (Models C, D, and E); the results are shown in Table 9.

The regression coefficients obtained with fuel volatility expressed as a percent evaporated are of opposite sign to those with fuel volatility expressed as the distillation temperature. This was expected because increased fuel volatility results in increased percent evaporated at a given temperature, but a reduced temperature for a given percent evaporated. The  $\mathbf{r}^2$  and coefficient of variation for these equations were no better than those based on distillation temperature. These equations also indicate the increased importance of the front-end volatility on driveability at low temperature.

# 5. Equations with Higher Order Terms to Improve Prediction

It was noted during earlier discussion that the simple expression for fuel volatility (see equation 2) did not adequately predict driveability performance of all fuels. To determine the effectiveness of the overall equation presented earlier (equation 8), predicted total weighted demerits were computed for each of the nine fuels used in this study. Demerits were computed at two temperatures, 5.5 and 51.3°F (the average run temperatures for the low and intermediate test phases, respectively). These calculated TWD are compared to average actual TWD in Table 11 and Figure 11.

At low temperature there were 2 points and at intermediate temperature there were 3 points which substantially deviated from a linear least-squares line drawn through the data points. All of the points showing the relatively large deviation represent fuels with nominally parallel distillations (Fuels 1, 2, and 3). It was of some concern that the predicted demerits for these fuels did not closely correspond to the actual demerits because these fuels more closely represent commercial fuels than do the other 6.

The fuel showing the largest deviation was Fuel 2, the midpoint of the fuel design cube (Figure 1). This indicated that driveability performance was not a linear function of the distillation temperatures. Therefore, additional regression analyses were conducted with nonlinear terms involving the distillation temperatures.

A stepwise regression was run starting with the 8-coefficient model (equation 8) and adding additional nonlinear terms one at a time. The 8-coefficient equation was used as a starting point to insure that the

final equation would not contain higher order effects for which the corresponding first-order effects were missing. The regression routine was given the opportunity to select either of the squares of the distillation temperatures  $(T_{10}^2, T_{50}^2, T_{90}^2)$  or the interaction terms  $(T_{10} \cdot T_{50}, T_{10} \cdot T_{90}, T_{50} \cdot T_{90})$ . When the model was restricted to two additional terms (which was reasonable based on the regression results), addition of the terms,  $T_{10} \cdot T_{50}$  and  $T_{50}^2$ , provided the greatest improvement in fit. The resulting 10-coefficient regression model is shown in Table 9 (Model F).

The improvement in prediction ability for the 10 coefficient model is shown in Table 11 and in Figure 12. With the 10-coefficient model, residuals (deviations between predicted and actual demerits) for Fuels 1, 2, and 3 were substantially smaller than those for Fuels 1, 2, and 3 with the 8-coefficient model. Furthermore, residuals with the 10 coefficient model for Fuels 1, 2, and 3 were similar to those for the other 6 fuels.

The utility of this expanded model would be somewhat limited however, if it applied only to these 1980 model cars. Using the data for 1973 cars<sup>1</sup>, the following regression equation, which includes  $T_{10} \cdot T_{50}$  and  $T_{50}^2$  terms, was derived:

$$TWD = 593 + 2.99 \cdot T_{10} - 9.18 \cdot T_{50} + 0.443 \cdot T_{90} - 0.0108 \cdot T_{10} \cdot T_{50} + 0.0278 \cdot T_{50}^{2}$$
(9)

Figure 13 shows a plot of actual demerits versus demerits predicted according to equation 9 for 1973, 1975, and 1977 models. For each of these studies, addition of the two nonlinear terms improve the fit of the data by the equation. The improvement is particularly apparent for the fuels with parallel distillation curves.

D. CLASSIFICATION OF TOTAL WEIGHTED DEMERITS BY TEST CYCLE AND MALFUNCTION

Total weighted demerits provide an overall indication of driveability during the test, but give no information on the types of driving problems encountered, or when they occurred. In this section, demerits are compared for various portions of the test (called test cycles) and for various types of malfunctions. The rater adjustment factors developed for TWD were not applicable to demerits for each test cycle and malfunction, so no rater corrections were applied.

The demerits for individual test cycles were totaled in a manner similar to that used for TWD: within a given maneuver only the malfunctions giving the highest demerits per maneuver were counted. For the demerit classification by malfunction, however, all weighted demerits were counted. Consequently, the sum of all demerits by malfunction will generally be greater than the corresponding TWD value.

# 1. Demerit Accumulation by Test Cycle

To show the rate of accumulation of demerits, the test was divided into 7 parts: the initial start-up and idle period, and each of the 6 driving cycles. Demerits for Fuels 1, 2, and 3 are shown for each test part in Figure 14.

In general, the bulk of the demerits occur during the first 3 driving cycles following the start-up. During start-up and near the end of the test, the number of demerits was low. There were, however, apparent differences between the low and intermediate temperature phases in the distribution of demerits throughout the test. At low compared to intermediate temperature, for example, a proportionately larger number of demerits occurred during start-up, but a smaller number of demerits occurred during cycles 4, 5, and 6. Also, Figure 14 indicates that at lower temperature, cycle 1 demerits were substantially greater than cycle 2 demerits, while at intermediate temperature, cycle 1 and 2 demerits were roughly equal.

Differences among the 3 fuels were generally small. The relatively large percentages for Fuel 3 during cycle 1 at low temperature and for Fuel 1 during cycle 2 at intermediate temperature were apparent exceptions.

# 2. Demerit Classification by Malfunction

Demerits by malfunction and the total for all malfunctions, averaged across cars, are listed in Table 12. Table 13 shows the same data expressed as a percent of the total. The mean data for all fuels indicate that hesitations, stumbles, and acceleration stalls account for about 80% of the total demerits for each test phase. The percentage of demerits for acceleration stalls at low temperature (35.8%) was much greater than that at intermediate temperature (18.4%). Demerits for starting time and idle stalls were also greater at low than at intermediate temperature. The results for Fuels 1, 2, and 3 in Table 12 indicate that decreasing fuel volatility increases demerits for all malfunctions except idle roughness and surge.

# E. EVALUATION OF ALTERNATIVE MEANS FOR TOTALING AND WEIGHTING DEMERITS

The CRC method for totaling and weighting demerits to obtain TWD was established during the analysis of the data for the 1972 program.

Although this system has been used for all CRC data obtained since 1972, various alternative systems have been proposed. To evaluate the merits of some of these alternative systems, regression equations (to predict driveability as a function of fuel volatility and ambient temperature) were developed for the following alternative demerit counting systems:

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- Normal weighting factors, but no restrictions on totaling demerits for multiple malfunctions within a maneuver.
- Normal weighting factors and restricted totaling of demerits, but with 32 demerits assigned to each deceleration stall.
- 3. Weighting factors developed during a study of customer perception of driveability with demerit totaling restricted to the one malfunction giving the highest demerits per maneuver.

No rater corrections were applied to these data.

The effect of these alternative demerit systems was evaluated by developing regression equations (Model B from Table 9) for each of the alternative methods. Results of these regressions are compared in Table 14 to those with the traditional demerit scale without rater adjustments. (Note that the equation for normal raw TWD counting was quite similar to the one presented earlier for adjusted TWD.) The equation obtained for demerits totaled without restrictions on multiple malfunctions within a maneuver was similar to that with the traditional demerit totaling; however,  $r^2$  decreased and the coefficient of variation increased when the demerit totaling restrictions were removed. Inclusion of deceleration stall demerits had little effect on either the regression coefficients,  $r^2$ , or coefficient of variation.

The regression equation obtained using the customer weighting factors was somewhat different from that for the normal TWD calculations. Regressions based on the customer scale, compared to the normal scale, gave a larger r2, but also a larger coefficient of variation. With the regression coefficients listed in Table 14 it is difficult to compare the relative importance of  $T_{10}$ ,  $T_{50}$ , and  $T_{90}$  for the normal and customer demerit scales. To allow such a comparison, the average ambient temperatures for the low and intermediate temperature phases were substituted for ambient temperature in both of the models, and the results are shown in Table 15. Regression equations for both demerit weighting schemes predict that, among the 3 distillation temperatures, Tio has the largest influence on driveability at low temperature and Tio has the largest influence on driveability at intermediate temperature. The major differences between the two demerit scales occur at intermediate temperature, where the customer scale indicates increased importance of Tio and decreased importance of Tio relative to the traditional scale.

Use of the customer scale results in a regression model similar to that of the traditional scale, but the differences between the two appear sufficiently large to warrant further evaluation of the customer scale in future programs.

# VI. REFERENCES

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TABLES

AND
FIGURES

TABLE 1. DESCRIPTION OF TEST CARS (1980 NODEL YEAR)

OLD TO	222	W.4.	4 K	4.4 N.N.	4.4 ~ ~	ww bw	2.0	2.0
MANIF VACU LT	11.0	10.5 8.0	11.0	12.0 8.5	8.0 10.0	8.0	8 2	5.0
VIN	4H69AAH164023 13689AY147578 1B680AY136618	1X685A6121354 1X085AW154471	1X657A6168393 3X37RAX121297	3116 9RAX112222 0K93D134123	0K92D117640 0T10A123994	0720A605259 0264G607627	NL41CAB119873 ZL44AAD159366	ML24AAD189995
AIR CORD.	YES NO YES	YES	YES	YES	YES	YES	28	YES
AIR INJECTION	YES PULSE PULSE	NO	PULSE NO	YES	YES	YES	YES	2
CLOSED	YES YES NO	YES	YES	NO YES	XO Xes	NO	YES	2
ENISSION	CALIFORNIA CALIFORNIA 49-STATE	CALIFORNIA 49-STATE	CALIFORNIA	49-STATE CALIFORNIA	49-STATE	49-STATE 50-STATE	CALIFORNIA	49-STATE
CARB. BBLS.	888	200	NJ	40	~~	100	~	۰
SHGINE MISP, L	89.5	2.5	5. 8.	4.2	4.2 5.2		3.7	1.7
HODEL	CENTURY CHEVETTE CHEVETTE	CITATION	CITATION DELTA-88	DELTA-88 Fairfyort	FAIRMONT	BOBCAT	ASPEN	HORIZON
MAKE	BUICK CHEVROLET CHEVROLET	CHEVROLET	CHEVROLET OLDSHOBILE	OLDSMOBILE FORD	FORD	MERCURY	DODGE	PLYMOUTH

NOTE: THE PLYNOUTH HORIZON HAD A MANUAL TRANSILSSION. ALL OTHER CARS HAD AUTOMATIC TRANSMISSIONS. \*MANIFOLD VACUUM (IN. HG) AT WHICH LIGHT THROTTLE (LT) AND DETENT (DT) ACCELERATIONS WERE PERFORMED

TABLE 2. IDENTIFICATION OF CARS ACCORDING TO FUEL SYSTEM AND CRC CODE

GRC CODE	01223   ** N1223   118457   118457   118457   118457   118457   118457   118453   118453   1184258   1184258   1184258   118453   1184258   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453   118453
FUEL SYSTEM	CLOSED LOOP OPEN LOOP CLOSED LOOP OPEN LOOP
CAR NO.	

\*BRACKETS INDICATE MATCHED PAIRS OF CLOSED-LOOP AND OPEN-LOOP CARS.

TABLE 3. DESCRIPTION OF TEST FUELS

				FUE	FUEL MUMBER	~			
		2	2	4	2	9	7	•	6
API GRAVITY SPECIFIC GRAVITY (60°F)	50.6	60.1	67.1	57.3	53.5	50.2	64.5	52.3 0.770	56.6
BISTILLATION (D-86)	& 6.	11.2	13.6	13.1	9.5	9.5	13.1	e0 •0	13.4
<b>.</b>	90	84	80	100	95	92	80 86	91	105
20% EVAP.	163	135	115	122	141	175	113	146	130
SON EVAP.	214	189	159	192	174	223	156	179	203
50% EVAP.	237	219	187	238	193	239 256	193	198 226	245 241
70% EVAP.	295	266	234	294	229	268	270	272	273
80% EVAP.	339	301	260	321	248	283	319	318	282
90% EVAP.	362	328	308	\$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$	296	303	356	356	294
EP EVAF.	408	373	370	402	367	360	406	398	359
DISTILLATION, % EVAP. AT:	ı	;		:	•		i	•	:
13505	102	11 20 20	200	17 25	۱ ا	n er j	32	9 5 1	1 2 2 3
158 <f 215°F</f 	\$ <b>0</b>	% & V & V &	9 0	4 7 7	61	36		22	625
230 oF	8 5 5	τυ « τυ ∈	<b>€</b> 0 €	& r & c	Z 5	ক <b>ত</b>	60 7	61	4 5 5
a cons	282	91	926	82	95	97	<b>8</b>	2 Z	
TEMPERATURE (°F) AT 20 VAPOR-LIQUID RATIO	146	128	111	121	133	147	116	140	121

0 2 0 0 2 0 0 0

MIXED

Fue i:

TABLE 4. TEST DESIGN

2/18 23 4	0000000
2/15 22 3 3	ပေရစပေမစ
2/14 21 5	0 6 6 0 0 6 6 1
2/13 20 9	02200220
2/12 19 8	0 8 8 0 0 8 9 0
2/11 18 6 6	0000000
/10 17 7	
-	0 2 2 0 0 2 2 0
	0000000
22	A R A R A R A R A A A A A A A A A A A A
2/6 13 [XEB	44 8 4 8 4 8 4 8 4 8 4 8 4 8 4 8 4 8 4
2/5 12 18	< * < * < * < *
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<b>⊣</b>	*****
:. •	
	<u>:</u>
	2/12 2/13 2/14 2/15 19 20 21 22 8 9 5 3

Abstorn designate traimed raters.

TABLE 4. TEST DESIGN (CONTINUED)

Test Design Intermediate Temperature Phase

5/14 26 1	Z Z Z Z Z	I
5/13 25 <u>5</u>		I
5/12 24 1	HEHEHEHE (V) HEHEHEM	z
5/10 23 <u>7</u>	2	
22 4		
5/8 21 6		
5/7 20 2		=
3/6		
5/4 18 1		
5/3 17		
5/2 16 8		
5/1 15 2	<b>工作工作工作。                                </b>	<b>z</b>
4/30 14 7	0 A A U O A A O O A A O O A A O O A A O O A A O O A A O O O A A O O O O O O O O O O O O O O O O O O O O	u
4/29 13 3	capopaco w copeche	Ľ
4/28 12 9	0 A A D O A A D O O A A D O O A A D O A A D O A A D O A A D O A A D O A A D O A A D O A A D O A A D O A A D O	e e
4/26 11 2	0000000 A: 0000000	Ŀ
4/25 10 1	0200200 7: 022002	Ľ
4/24		င
4/23 8 5	000000 ~ 000000	ပ
4/22 7 3	000000 N 000000	v
4/21 6 1	とわりととりいと らっ ドドリリとうり	
4/19 5 6	そひりきどりりき 今 きどりりぎりり	W.
4/16 4/17 4/18 2 3 4 4 8 8 8	ырымары wi жыромар:	P)
4/17 3 9		₽
4/16 2 1	<b>88888888 (1) 88888888</b>	<b>L</b>
1 1 2 2	<b>№ 00 m m 00 m 00 m 00 m 00 m 00 m 00 m </b>	
Date: Test Day: Fuel:		
	Ca 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	£

\* Letters designate traimed raters.

TABLE 5. DRIVING PROCEDURE FOR THE MANUAL TRANSMISSION VCHICLE

MANUEVER	PROCEDURE
0-25 mph light throttle acceleration	start 1st gearshift to 2nd at 15 mph
25 mph cruise	shift to 3rd gearcruise
25-35 mph detent acceleration	acceleration in 3rd gear
0-35 mph WOT acceleration	start 1st gearshift to 2nd at 25 mph shift to 3rd at 35 mph
10-25 mph light throttle acceleration	2nd gear
idle	in neutral
0-45 mph crowd acceleration	start 1st gearshift to second at 15 mph shift to 3rd at 25 mph shift to 4th at 42 mph

TABLE 6. ADJUSTED TOTAL WEIGHTED DEMERITS SUMMARY

	TIEAN	42.	_	21.	٠	57.	ş.	97.	6.3			3 6	5 (	9		72.	eo		132.3		MEAN	17.		۶.				· ~	33.	44.	66	'n.	525	ė.	55.9	124.5
	FUL 9	ς.	55.	<u>"</u>	m.	ë	95.	41.	•				<u>:</u> :	9	5	92.	13.	10	142.0		FUFL 9	32		•	٠,				51.	53.	02	99.	. 6	7 6	38.2	130.0
	FUEL 8	56.	16.	16.	•	79.	56.	29.	52	. 6			2 :		2	<u>.</u>			137.5		FUEL 8	26.	 •	•	_; _		,		62.	69.	95	90	5.	٠,	70.3	120.7
	FUEL_1	9	02.	ς.	•	ŝ	67.	55.	e C				• •	×	٠	∹	_	5.0	82.6	SE	FUEL 7	Į,		mi.	٠	, ,			99.	22.	9	79.	31.	??	\$ C	102.4
E PHAS:	FUEL 6	18.	g.	Ę5	•	90	39.	13.	96				9	D		95.	د	44.5	191.5	ATURE PHA	FUEL 6	36		58.	13.	? ~		37.	82.	77.	97	14	. 6	<u>;</u> ;	55	150.9
TEMPERATURE	FUEL 5		6	93		41.	'n	6	P.	77			o	9	_;	65	S.		128.2	ATE TEMPER	FUEL 5	ď	, r			67			85.	_;	6	_;	<u>.</u>	v.	41.3	89.5
101	FUEL 4	9	83.	_	80.	6		_	«					56.	ς.	72.	€.	-11.0	119.3	INTERMEDI	FUEL 4	17	: ::	64.	39.			٠,	45	84.	36.	ဗ	78.	9	136.0 71.2	156.6
	FUEL 3	5.	۲,	Ŗ,	,	9	€0	_		•	? •	0 :	٠,	48.	÷	m.	0	-10.5	73.6		FUEL 3		'n	_;		> c • •	v ~	m	53.	ö	٠	73.			28.2 28.2	77.6
	FUEL 2	24.	9	9		2	•	<b>_</b>			٠.	9	∹,	5	ç	74.	€.		108.4		FUEL 2	9	: :	_;	ς,	• •		'n	31.	35.	6.	86	60	0	31.2	103.3
	FUEL 1	15.	14.	55.	_	12.	66	6				<u>`</u> ;	÷:	۵.	54.		54.	15	207.7		FUEL 1	2		05.	. 62		9 4	55.	30.	82.	<b>6</b> 0	34	8	. 2	78.0	189.8
	CAR	-	7	~	•	'n	•		• •	9 0								16	MEAK		CAR		• ~	<b></b>	<b>~</b> 1	n 4	•	- •0	•						16	MEAN

TABLE 7. ANALYSIS OF VARIANCE TO DETERMINE SIGNIFICANCE OF CAR, FUEL, AND TEST PHASE EFFECTS

1501C2	70	SQUARES	SOUPES	-	CONFIDENCE I EVEL, 3
	<u>-</u>	2675883	178392	87.6	6.66
CAR	0.1			8	00
FUEL	€0	1215992	151999	0	
TEST PHASE	-	7839	7399	0.5	37
CAR BY FUEL	120	379743	3164	1.6	6.66
CAR BY TEST PHASE	15	489872	32653	16.0	6.66
FUEL BY TEST PHASE	••	92129	11591	5.3	6.66
CAR BY FUEL BY TEST PHASE	120	264170	2201	1.1	7.1
ERROR	530	1079480	2037		

TABLE 8. ANALYSIS OF VARIANCE TO DETERMINE SIGNIFICANCE OF FUEL SYSTEM (OPEN VS. CLOSED LOOP) EFFECTS USING DATA FOR MATCHED CARS ONLY.

SOURCE	J.	SUN OF	MEAN Squares	ía.	CONFIDENCE LEVEL, 2
SYSTEM	***	150	150	0.0	~
CAR	10	951864	95183	47.4	6.66
FUEL	••	955129	110391	45.7	6.66
TEST PHASE	-	8343	8343	4.0	55
SYSTEM BY FUEL	•	13211	1651	9.0	25
SYSTEM BY TEST PHASE	-	37246	37246	1.6	7.7
FUEL BY TEST PHASE	•	63821	7978	3.7	6.66
FUEL BY CAR	80	208977	2612	47.4	56
CAR BY TEST PHASE	10	227524	22752	11.3	6.96
SYSTEM BY FUEL BY TEST PHASE	••	13546	1693	0.8	38
CAR BY FUEL BY TEST PHASE	80	172262	2153	1.1	99
ERROR	397	798058	2010		

TABLE 9. SUMMARY OF REGRESSION ANALYSES WITH ALTERNATIVE DEMERIT SYSTEMS

ATE TEMPERATURE: 386.5 0.6664 1.1285 0.5770 386.5 0.6664 1.1285 0.5770  88.8  RATURE: 351.3 1.9887 0.9631 0.1364*  351.7 2.0532 0.9925 0.1576* 0.02517 0.002245* 0.007379 312.0 -5.0729 -0.1780* -0.2592* 1.6533* 0.065078 -0.0025479 312.0 -5.0729 -1.8981 -0.3535* 1.1278* 0.065078 -0.003410* -0.025417* 348.4 -4.1290 -1.8981 -0.3552* 0.069189 -0.003410* -0.025417* 378.8 -4.3014 -2.5518 -0.7708* 1.6352* 0.069189 0.002385* 0.007320 0.010321* 0.044768 62.1	норет ро	b1	b2	b3	مً	p.	bę	b,	ba	6q		:
0.9631 0.1364*  0.9631 0.1576* 0.0251* -0.027171 0.002245* 0.0025779  -0.1780* -0.2592* 1.6533* 0.063541 -0.028659 -0.025779  -1.8981 -0.3535* 1.1278* 0.050078 -0.007046* -0.025620*  -2.5518 -0.7708* 1.6352* 0.069189 -0.003410* -0.025417*  19.5514 0.2949 0.0759* -0.027829 0.002385* 0.007320 0.010321* 0.044768 62.1	ERMEDIATE TE	:NPERATURE: 0.6664	1.1285	0.5770							49.3	0.22
0.9925 0.1576* 0.0251* -0.027171 0.002245* 0.0025779 62.6 62.8 -0.1780* -0.2592* 1.6533* 0.063541 -0.028659 -0.025779 62.8 -1.8981 -0.3535* 1.1278* 0.050078 -0.007046* -0.023620* 62.9 -2.5518 -0.7708* 1.6352* 0.069189 -0.003410* -0.025417* 0.044768 62.8 -19.5514 0.2949 0.0759* -0.027829 0.002388* 0.007320 0.010321* 0.044768 62.1	TEMPERATURE - 351.3	1.9887		0.1364*							68.8	0.17
	18 INED: 371.7 312.0 348.4 578.8 1894.6	2.0532 -5.0729 -4.1290 -4.3014 -0.0353*	0.992 -0.178 -1.898 -2.551 -19.551	0.1576* -0.2592* -0.3535* -0.7708* 0.2949	0.0251* - 1.633* 1.1278* 1.6352* 0.0759* -	0.027171 0.063541 0.050078 0.069189	0.002245* -0.028659 -0.003410* -0.003410*	0.005579 -0.0255779 -0.025620* -0.025417*	0.010321*	0.044768	62.9 62.9 62.3 62.3	0.17 0.16 0.16 0.18

\*\* COEFFICIENT OF VARIATION (STANDARD ERROR EXPRESSED

NOTE: THE LOW 12 VALUES RESULT FROM LARGE CAR-TO-CAR VARIATIONS, WHICH ARE IN THE ERROR TERMS OF THESE MODELS.

TABLE 10. SUMMARY OF REGRESSION ANALYSES TO DETERMINE CAR EFFECTS

b <sub>6</sub> b <sub>7</sub> COV** r <sup>2</sup>	012608* 0.004529* 25.8 0.65 001171* 0.013596* 29.9 0.57 006686* 0.0002246* 50.0 0.57 0060374* 0.006874* 58.6 0.35 003028* 0.014265* 35.0 0.63 0017326* 0.01971* 21.9 0.51 001429* 0.01971* 37.6 0.62 001029* 0.01975* 37.6 0.62 001029* 0.010239* 45.2 0.46 001029* 0.003529* 35.8 0.40 005497* 0.003529* 35.8 0.40 005505* 0.016703 97.7 0.48 0005517* 0.00337* 56.8 0.33
bs 1	013112 × 0 0103356 × 0 010371 × -0 0120538 × 0 067855 0 013057 × -0 013057 × -0 053136 -0 053138 -0 053138 -0 07237 × -0 07237 × -0 07237 × -0 07237 × -0 07353 × -0 073737 × -0 073737 × -0 073737 × -0 073737 × -0 073737 × -0
p,	-3.2162* 2.6253* 3.47652* 3.47652* -5.3087* -6.8099* -6.8099* -1.215 -1.8829* -1.8829* -1.8829* -1.8829* -1.8829* -1.8829* -1.8829* -1.8829* -1.8829* -1.8829* -1.8829* -1.28810* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868* -1.2868*
Ъэ	0.06233338 0.05339338 0.05339338 0.05339338 0.05339338 0.05339338 0.0533938 0.0633838 0.0633838 0.0633838 0.0633838 0.0633838 0.0633838 0.0633838 0.0633838 0.0633838 0.0633838 0.0633838 0.0633838 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.06338 0.
, p	0.7745 0.79465 0.7855 0.7855 0.7855 0.5855 0.8139 0.6557 0.6557 0.1638 0.1638 1.0934 1.1120
b <sub>t</sub>	1.4947 1.9027 1.9027 1.34423 1.34423 4.0474 4.0474 1.7934 1.7934 1.7934 0.95860* 0.7266 1.7266
bo	
CAR	1 2 4 4 5 6 6 7 7 10 11 12 13 14 16 0PEH LOOP CLOSED LOOP

MODEL: TWD = bo + b1 • T10 + b2 • T50 + b3 • T90 + b4 • Ta + b5 • Ta • T10 + b6 • Ta • T50 + b7 • Ta • T90

\* INDICATES COEFFICIENTS NOT SIGNIFICANT AT A 90% CONFIDENCE LEVEL. \*\* COEFFICIENT OF VARIATION (STANDARD ERROR EXPRESSED AS A PERCENT OF THE NEAN).

TABLE 11. COMPARISON OF ACTUAL TO PREDICTED DEMERITS USING THE 8-COEFFICIENT AND 10-COEFFICIENT MODELS

TABLE 12. CLASSIFICATION OF DENERITS BY NALFUNCTION

	236.3 124.0				151.6			TOTAL	220.0	103.5	171.7	93.1	103.3	123.6	ן אַנו	1.	
ACCELERATION STALLS	100.5 33.0			56.0	54.2		oces EDATION	STALLS	~	000	<i>\$</i> 60	12.0	ΛĐ	23.0	,	0.47	
AC BACKFIRE	6.0 0.0	V.0.	×. ×. ×.	22.5	2.4		•	BACKFIRE	7 7	. N.	٠. د د. ه	1.7	6.9		1:1	9. P	
311.75	4.4	2.4 1.5	೧.ಗ. ಐ.ಸ.	21.9	2 7		IVSE	SURGE			<b>4.</b>	9.9	4. u	. W.	o •	5.6	
PHASE	39.7	20.5	26.9 37.3	17.3 24.1 40.1		1.82	ERATURE PH	STUMBLE		25.4	29.5	52.1 27.4	58.5	30.8	45.6	8.04	
<u>LOU TEMPERATURE PHASE</u> E IS <u>HESTIATION</u> STUMB	55.7	200 x	35.45 5.45 5.45 5.45 5.45 5.45 5.45 5.45	80 6 4 80 6 4 80 6 4	) 1 - (	39.7	INTERNEDIATE TEMPERATURE PHASE	HESITATION		4.79	23.2	56.1	41.7	28.4 44.6	47.8	41.9	
IDLE STALES	16.5	D.T.F		7.7.	T . 0 T	ಜ.	INTER	IDLE	244116			•		4.00	•	2.1	
IDLE	8.9			. 10 V.		9.9		IDLE	KUDRUNGSS		ก่ง	. w	, , , , , , , , , , , , , , , , , , ,	14.0	าพา	13.7	
START	10.7	8.5	2.7	1.80 8.50 1.00 1.00 1.00 1.00 1.00 1.00 1.00 1	12.3	9.6		START	LINE	2.1	U	) 4.	M.C		2.9	1.7	ı
į	1	1 67 KY	÷ល	9 <b>~</b> 80	o	NEAN			FUEL	-	· 8 ·	n 4	· LO •	o <b>~</b> !	10 P	MEAN	

NOTE: TOTAL EXCEEDS TWD (TABLE 6) BECAUSE ALL MALFUNCTIONS ARE COUNTED.

TABLE 13. CLASSIFICATION OF DEMERITS BY MALFUNCTION, %

2011 AC 2014 AC 201	LOW TEMFERATURE FUASE	IDLE IDLE ROCELERATION STUMBLE SURGE BACKFIRE STALLS TOTA	2.9 6.1 23.6 16.8 1.9 1.7 42.5 100	4.3 5.6 31.8 16.6 1.7 1.6 50.7 100.	7.1 4.6 32.2 22.9 2.7 0.8 53.6 100.	5.3 8.0 27.6 19.2 1.1 2.9 50.1 100.	4.1 4.0 24.2 13.7 2.0 0.0 41.0 100.	1.7 4.6 25.5 17.5 2.5 1.3 37.6 100.	5.6 7.2 36.4 17.4 1.9 3.1 19.1 100.	4.6 4.5 22.8 15.5 U./ 1.4 11.3 34.1 100.	2.9 2.4	2 4.3 5.6 26.2 16.6 1.8 1.6 35.8 100.0	THE PHASE	THE EXPEDIATE TOTAL CALL SALES	ART IDLE IDLE ACCELERATION STUMME SURGE BACKFIRE STALLS TOTAL	1000	6.7 2.1 30.7 25.2 2.5 2.5 2.7 100.	11.7 1.2 53.6 27.9 6.2 2.3 12.0 100.	18.1 0.3 30.0 37.7 5.9 2.2 3.6 100.	3.0 2.3 32.7 30.4 4.7 5.4 15.6 100.	14.6 0.3 33.6 29.4 7.1 1.0 12.7 100.	3.1 1.0 24.6 34.5 2.6 3.5 25.1 100.	13.6 2.3 27.5 37.1 5.0 3.0 7.2 100.	10.7	3 10 3 16 31.2 30.4 4.1 2.7 15.4 100.0
	ה		9 6 6	) KT	5	- M	~	5	5.00	9.5	4.2	4.3 5.			IDLE J ROUGHNESS SI		6.7 2.	11.7			14.6		13.6	10.7	7 07

TABLE 14. SUMMARY OF REGRESSION ANALYSES WITH ALTERNATIVE DEMERIT SYSTEMS

	p <sub>0</sub>	p.	p2	b <sub>3</sub>	ď	bş	b <sub>1</sub> b <sub>2</sub> b <sub>3</sub> b <sub>4</sub> b <sub>5</sub> b <sub>6</sub>	bγ	COV** r2	12
NORMAL (RAW) TUD	-390.5	2.0799 1.0118	1.0118	0.2142*	0.0045*	0.2142* 0.0045* -0.027419	0.003267*	0.006412* 61.7 0.19	61.7	0.19
NORMAL WEIGHTS, W/O RESTRICTIONS	-426.2	2.1532	2.1532 1.1281	0.2525*	-1.3969*	0.2525* -1.3969* -0.024277	0.005589*	0.005589* 0.008155* 68.2 0.16	68.2	0.16
SNITHNIOS CONT. ICHGON										
DECELERATION STALLS -430.1	-430.1	2.2444 1.0364	1.0364	0.2890	0.7056*	0.7056* -0.030773	0.003480*	0.004259* 61.0 0.20	61.0	0.20
CUSTOMER WEIGHTS	-805.3	3.3590	3.3590 1.8383	0.7210		5.3336 -0.043356	-0.010630* -0.000807* 77.2 0.29	-0.000807*	2.77	0.29
REGRESSION MODEL:	TWD = bo	+ b, T, 10	+ b2 • T50	+ b3 • T90	+ b, •Ta +	bs Ta Tio	o + b1. T10 + b2 T50 + b3 T90 + b4 Ta + b5 Ta T10 + b6 Ta T50 + b7 Ta T90	+ b, Ta T		

\* INDICATES COEFFICIENTS NOT SIGNIFICANT AT A 90% CONFIDENCE LEVEL. \*\* COEFFICIENT OF VARIATION (STANDARD ERROR EXPRESSED AS A PERCENT OF THE MEAN).

TABLE 15. COMPARISON OF NORMAL AND CUSTOMER DEMERIT SCALE USING NORMALIZED REGRESSION EQUATIONS

DENERIT	p <sub>0</sub>	bı	b <sub>2</sub>	b <sub>3</sub>	υ
TEMPERATU	LOW TEMPERATURE (5.5°F):				
NORMAL	-390.5	0.593	0.327	0.079	3,1455
CUSTOMER	-776.0	0.556	0.317	0.128	5.6169
CRNEDIATE	INTERNEDIATE TEMPERATURE (51.3°F)	(51.3°F)			
NORMAL	-390.3	0.281	0.492	0.226	2.3958
CUSTOMER	-531.7	0.365	0.416	0.219	3.1073

NODEL: TWD = bo + (b<sub>1</sub>•T<sub>10</sub> + b<sub>2</sub>•T<sub>50</sub> + b<sub>3</sub>•T<sub>90</sub>)c

NOTE: NO RATER ADJUSTMENTS WERE APPLIED TO THESE DATA.

PIGURE 1. LAYOUT OF TEST SITE

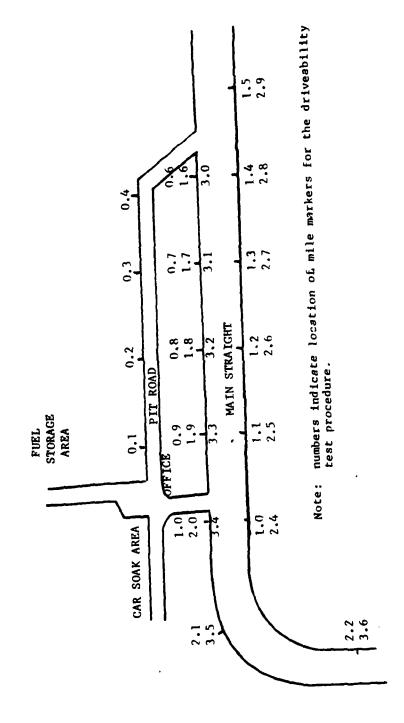


FIGURE 2. DISTRIBUTION OF AMBIENT TEMPERATURES DURING TEST RUNS

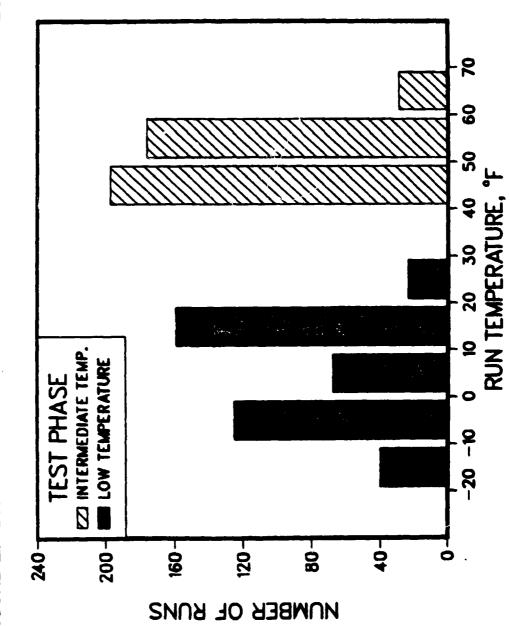


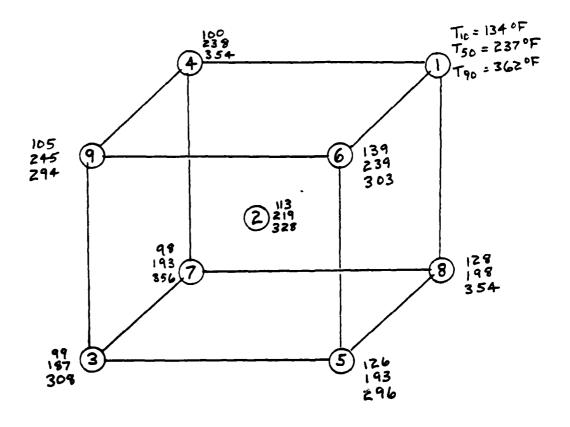


FIGURE 3. EFFORTS TO REMOVE ICE FROM TRACK DURING THE LOW TEMPERATURE PHASE



FIGURE 4. PHOTOGRAPH SHOWING TEST FUEL CAN LOCATION AND COLD STARTING ASSISTANCE

FIGURE 5. TEST FUEL DESIGN



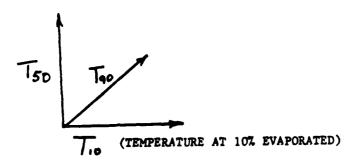
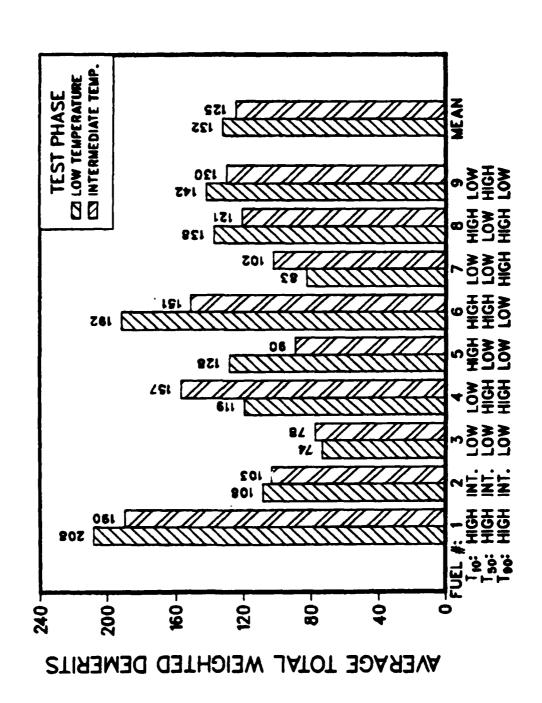
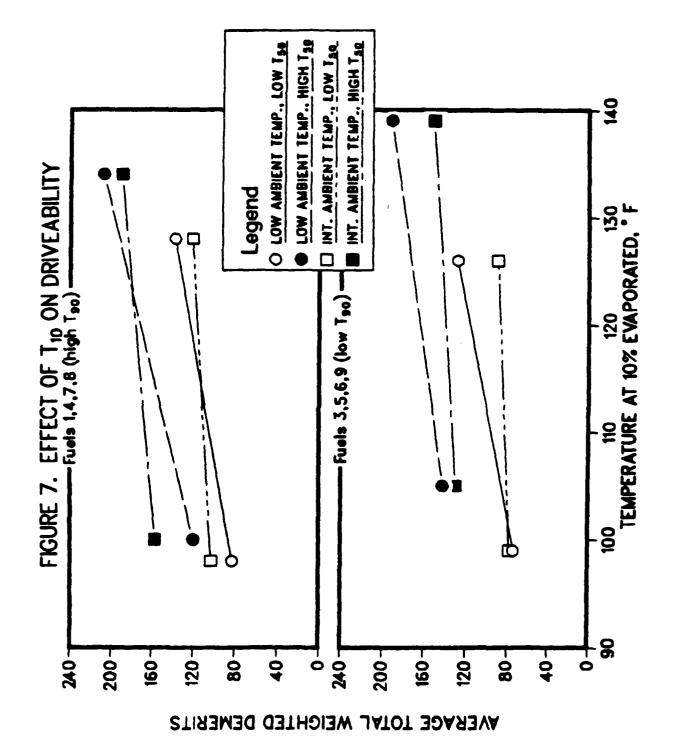
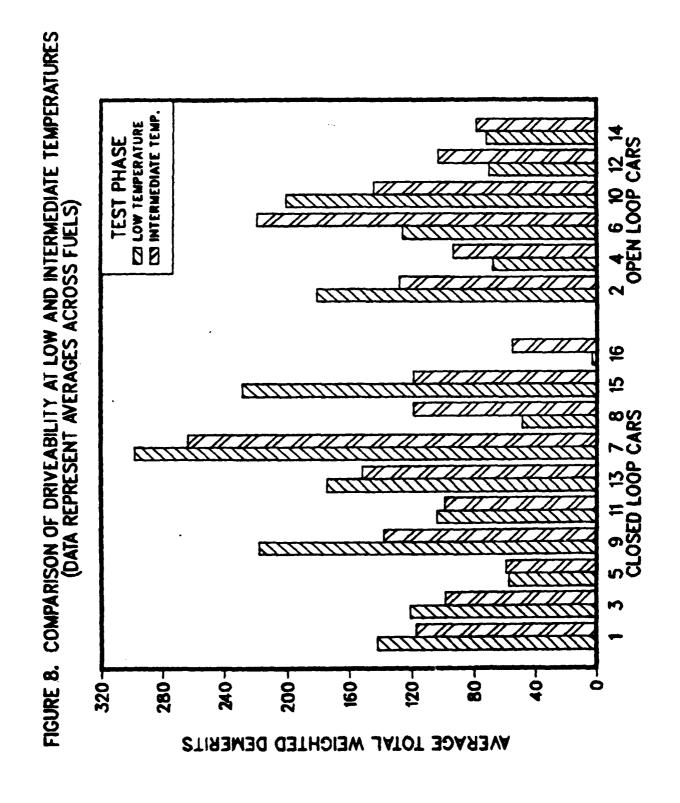
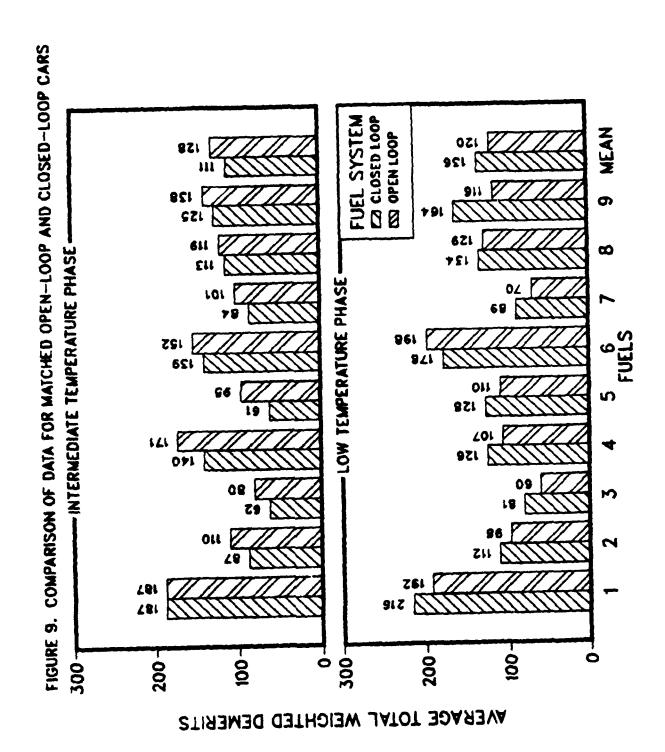


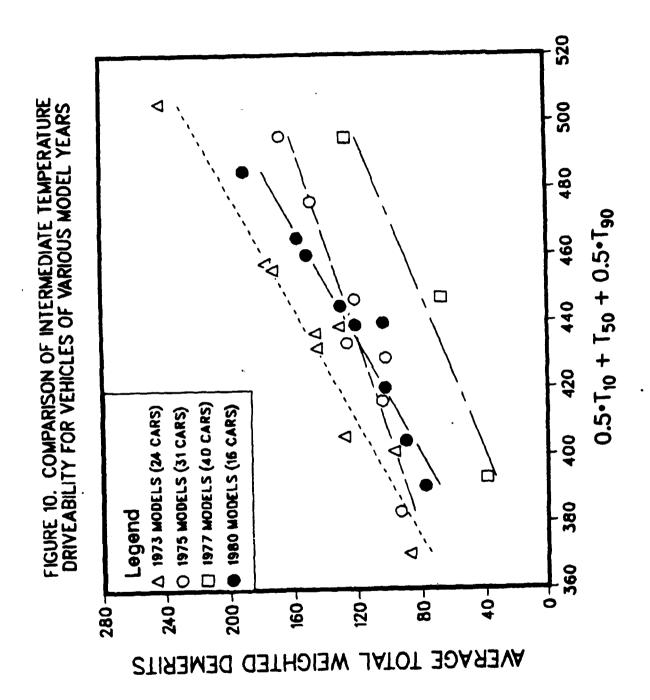
FIGURE 6. COMPARISON OF DRIVEABILITY AT LOW AND INTERMEDIATE TEMPERATURES (DATA REPRESENT AVERAGES ACROSS CARS)

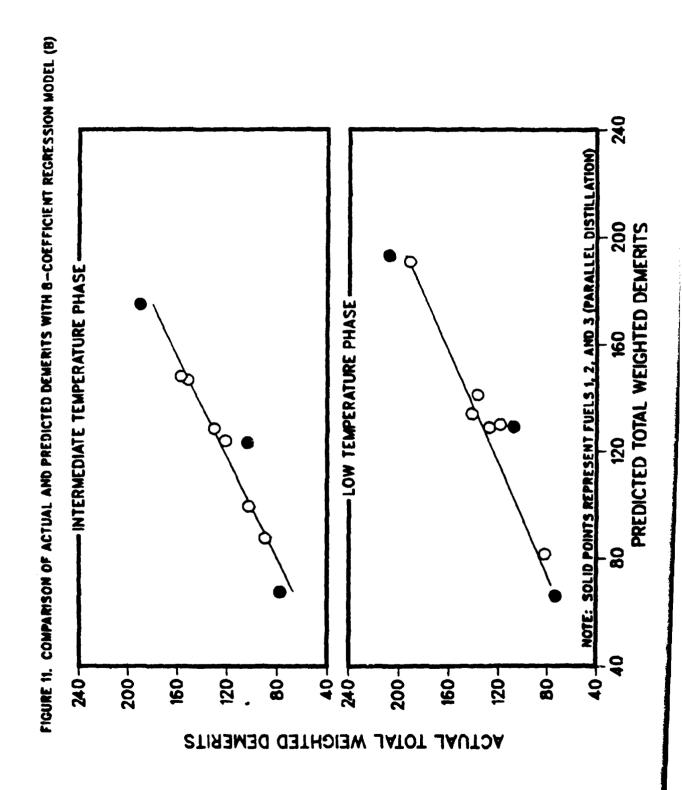












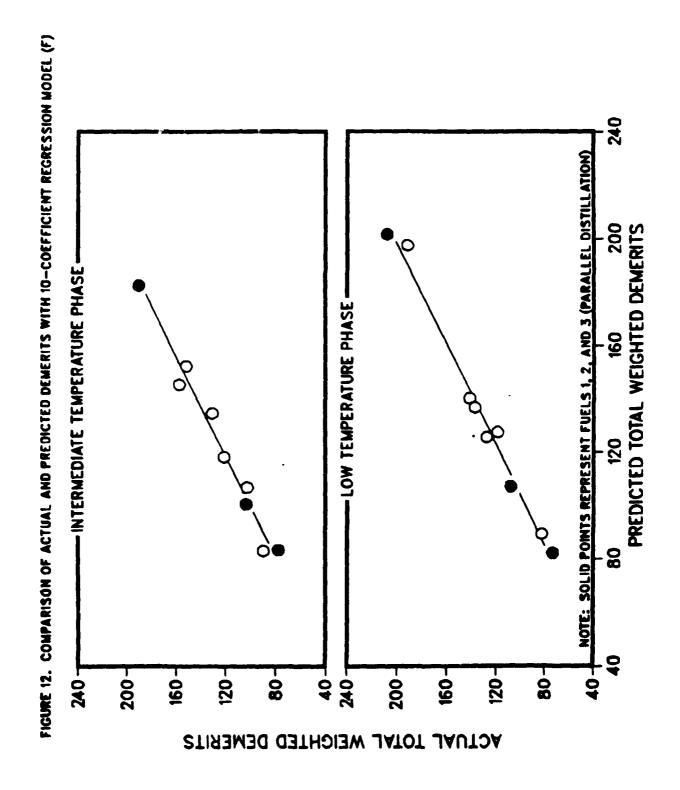
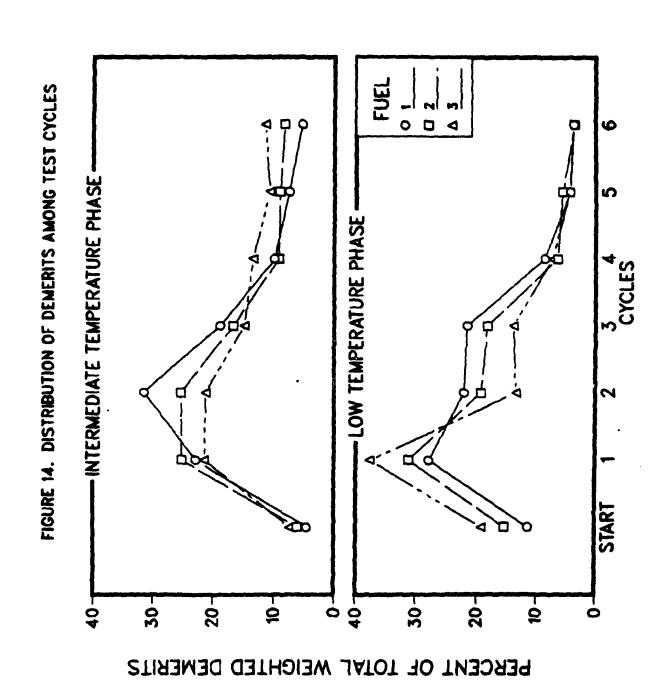


FIGURE 13. EVALUATION OF NONLINEAR EQUATION FOR PREVIOUS MODEL YEARS  $593 + 2.99 \cdot T_{10} - 9.18 \cdot T_{50} + 0.443 \cdot T_{90} - 0.0108 \cdot T_{10} \cdot T_{50} + 0.0278 \cdot T_{50}$ NOTE: SOLID POINTS REPRESENT FUELS 1, 2, AND 3 (PARALLEL DISTILLATION) 200 160 120 1977 MODELS (40 CARS) 240- | △ 1973 MODELS (24 CARS) O 1975 MODELS (31 CARS) Legend 200-40-80-120-160 280-TOTAL WEIGHTED DEMERITS **AUTDA** 



-55-

# APPENDIX A

PROGRAM PARTICIPANTS AND PANEL MEMBERSHIPS

# Table A-1. Program Participants

# Low Temperature Phase

#### Name

#### Company

<pre>C. E. (Charley) Baxter*</pre>	Mobil Research and Development
N. D. (Norm) Brinkman	General Motors Research Laboratories
G. (Gene) Ellis	Systems Control, Inc.
D. W. (Doug) Hall	Chevron Research Co.
A. (Al) Mallow	Systems Control, Inc.
R. V. (Vance) McCabe	General Motors Research Laboratories
J. D. (Jim) Merrit*	Amoco Oil Co.
W. K. (Bill) Okamoto*	Ford Motor Co.
R. (Dick) Schwartzlander	Gulf Research and Development Co.
J. K. (Ken) Slack*	Suntech, Inc.
F. J. (Fred) Villforth**	Texaco Inc
W. C. (Walt) Williams	Amoco Oil Co.

# Intermediate Temperature Phase

#### Name

# C. E. (Charley) Baxter\* D. W. (Doug) Hall\* G. S. (George) Hyek\* W. R. (Bill) Mallett\* J. E. (Jim) Robinson\*\* E. H. (El) Schanerberger\* E. D. (Lou) Steinke B. Y. (Brian) Taniguchi S. (Sam) Vallas

# \*Rater \*\*Coordinator

#### Company

Mobil Research and Development
Chevron Research Co.
Gulf Research and Development Co.
Union Oil Co.
Sohio
Ford Motor Co.
Suntech, Inc.
General Motors Research Laboratories
Amoco Oil Co.

Table A-2. Program Planning and Data Analysis Panel Memberships

	<u>Panel Me</u>	mbership	
Name	Planning	<u>Analysis</u> *	<u>Company</u>
N. D. Brinkman,	Ldr. X	x	General Motors Research Laboratories
J. H. Baudino	X	x	ARCO Petroleum Products Co.
C. Bello	X	X	Gulf Research and Development Co.
A. M. Horowitz		X	Mobil Research and Development
J. C. Ingamells	X	X	Chevron Research Co.
J. L. Keller	X		Union Oil Co.
W. R. Mallett		X	Union Oil Co.
C. R. Morgan	X		Mobil Research and Development
H. T. Niles	X	X	Ford Motor Co.
R. M. Reuter	X		Texaco Inc
J. E. Robinson	x	X	Sohio
R. T. Simmons	x		Chrysler Corporation
E. D. Steinke	x	X	Suntech, Inc.
F. J. Villforth		X	Texaco Inc
W. C. Williams	X	x	Amoco Oil Co.

<sup>\*</sup>The analysis panel appreciates the contributions of statisticians, G. S. Leithead and h. R. Crawford (Texaco) and J. W. Gorman (Amoco).

# APPENDIX B

PROGRAM PROPOSAL AND TEST PROCEDURE DETAILS

#### PROPOSED CRC PROGRAM

# Effect of Fuel Volatility on Driveability of 1980 Cars

## at Low and Intermediate Ambient Temperatures

#### Objective

To determine how gasoline volatility affects vehicle driveability at low  $(0-20^{\circ}F)$  and intermediate  $(40-60^{\circ}F)$  ambient temperatures. The test gasolines will be designed to allow independent estimation of front-end, mid-range and tail-end volatility effects, and the test cars will include those with and without closed-loop fuel control systems.

#### Introduction

Previously, several CRC programs have been conducted to determine the effect of fuel volatility on cold-start/warm-up driveability, but none of these test programs have been run at ambient temperatures below 30°F. A study of fuel volatility effects on low-temperature driveability is desirable in order to aid in the establishment of driveability specifications for fuels marketed at low ambient temperatures.

Starting with the 1980 model year, most cars sold in California are expected to use either a three-way catalyst or a reducing catalyst to control  $\mathrm{NO}_{\mathrm{X}}$  emissions. These catalysts will also probably be used on cars sold in all states beginning with the 1981 model year. To accommodate these  $\mathrm{NO}_{\mathrm{X}}$  catalysts, most cars will be equipped with exhaust gas oxygen sensors and closed-loop fuel control systems. These design changes will potentially have a major effect on driveability with various fuels.

This proposed program is designed to measure the effect of several fuel volatility parameters on driveability at low and intermediate ambient temperatures using cars with and without closed-loop fuel control systems. Tests at intermediate as well as at low temperatures are necessary due to the lack of volatility-driveability data for cars with closed-loop fuel control.

#### Program Summary

A cooperative, cold-start/warm-up driveability program will be conducted during 1980 at Donnybrooke Track near Brainerd, Minnesota. The program will be divided into two 4 1/2-week phases: low temperature tests from January 21 through February 21; and intermediate temperature tests from April 14 through May 15. Sixteen cars, ten with closed-loop and six with open-loop fuel systems, will be tested on nine gasolines designed to provide independent estimation of effects of front-end, mid-range, and tail-end volatility on driveability. The manpower requirement is about 54 man-weeks.

#### Test Temperatures

Target minimum test temperatures are 0°F (-18°C) and 40°F (4°C) for the low and intermediate temperature phases, respectively. Tests will be conducted as near as possible to these minimum target temperatures, however, due to daily ambient temperature fluctuations, actual ranges of test temperatures will probably be about 0-20°F (-18 to -7°C) for the low temperature phase and about 40-60°F (4-16°C) for the intermediate temperature phase.

#### Test Location and Timing

The Donnybrooke Track near Brainerd, Minnesota was chosen as the test site because it was the most suitable track found in an area with sufficiently low-ambient temperatures. The track layout is shown in Figure B-1. During late January and early February, ambient temperatures in Brainerd are suitable for the low-temperature phase of the program. A contractor will be hired to remove snow and ice from the track.

The Donnybrooke Track was also chosen for the intermediate-temperature phase of the program to avoid the expense of shipping test cars to an alternative site, and to avoid the potential influence of test track differences on the observed driveability results. Brainerd temperatures are suitable for intermediate temperature testing during late April and early May.

#### Test Cars

The tests will be conducted with 16 1980 model year cars. The test was limited to 16 cars because 16 was the maximum number which could be tested by the number of participants estimated to be on site at one time. If participation is sufficient, additional cars will be tested in one or both phases. To emphasize future emission hardware, ten of the cars will be equipped with closed-loop fuel systems and six cars with open-loop systems. In general, the closed-loop cars will be California cars and the open-loop cars will be 49-state cars. The ten closed-loop cars will be selected to represent the various types of available engine controls, and the six open-loop cars will be high production cars similar to six of the closed-loop.

Prior to the tests, the cars will be driven a minimum of 500 miles (800 km) and be inspected to insure proper mechanical operation.

#### Test Fuels

The test fuel design, given in Table B-1, is similar to that for previous CRC programs in which various parts of the distillation curve were varied independently. Fuel 1 and fuels 3 through 9 represent a complete factorial design for two levels of each of the three factors (10, 50, and 90 percent distillation temperature). Fuel 2 is an average volatility fuel included to check for nonlinearity.

The range for each of the three distillation points were chosen to represent the volatility of about 95 percent of commercial gasolines.

In addition, Reid vapor pressure limits have been specified to provide starting at low temperature, and freedom from vapor lock at intermediate temperature.

#### Test Procedure

The driveability test procedure will be the same as that used in past intermediate-temperature driveability programs. It consists of a cold start (after an overnight soak) followed by 3.6 miles of driving through various maneuvers such as light-throttle acceleration, cruise, detent acceleration, full-throttle acceleration, crowd acceleration, and idle. Starting time is measured and the number of stalls are recorded. Other malfunctions, such as hesitation, stumble, surge, idle roughness, and backfire, are rated subjectively by the driver on a scale of trace, moderate, or heavy. These malfunction observations are converted to demerits to indicate the severity of the driveability problems which were encountered during the test.

## Test Design

...

The test design for the low temperature phase is given in Table B-2 and for the intermediate temperature phase is given in Table B-3. The design provides duplicate tests on all fuels for each phase and additional tests on fuels 1 and 2 for rater comparison and rater repeatability determinations. Test results will be tabulated on site and additional tests will be scheduled, if necessary, to recheck outliers. For each phase there will be four raters with two on site at any given time. This allows the raters to participate in two-week rather than four-week shifts. To provide continuity, at least two of the raters must participate in both phases.

#### Program Duration and Manpower Requirement

As indicated in Table B-4, each test phase should last about 4 1/2 weeks. Dates selected for the low-temperature phase are January 21 through February 21, 1980, and the intermediate temperature phase from April 14 through May 15, 1980. No company will be expected to participate for the entire nine weeks of both phases. Six participants are required to be on site at all times; this gives a total manpower requirement of 54 man-weeks. Some companies will be asked to provide raters who participate for a total of 4 1/2 weeks (2 1/4 weeks per phase). Others could participate in 2 1/4 week shifts in one or both test phases. Efforts will be made to assign participants to the test phase in which they are most interested.

# Supplemental Investigation of Analytical Technique to Measure Fuel Vaporization Rate

Arco has developed a test method [Analytical Chemistry, Vol. 49, pp. 2368-2371 (1977)] to measure the rate of fuel vaporiation under conditions simulating those in an intake manifold. Arco has developed linear relationships between results with this method and driveability demerit results from past CRC programs. As part of their participation in this program, Arco will use this test method to measure vaporization

rates of the test gasolines, and attempt to correlate these measurements with driveability, as measured during both the low and intermediate test phases.

TABLE B-1

Fuel Specifications

Fuel No.	110	10%*	Distil] 30%**	Distillation Temperatures, °F (°C) 30%** 50%* 70%**	peratures,	%*%0L 10%**	***206	Reid Vapor Pressure, psi (kPa)
7	130	130 (54.5)	195 (90.5)		240 (115.5)	290 (143.5)	360 (182.0)	9.0-11.0 (62-76)
2	110	(43.5)	165 (74.0)		215 (101.5)	265 (129.5)	330 (165.5)	10.5-12.5 (72-86)
3	06	90 (32.0)	135 (57.0)	(88.0)	88.0)	230 (110.0)	300 (149.0)	12.0-14.0 (83-97)
4	06	90 (32.0)	155 (68.5)		240 (115.5)	290 (143.5)	360 (182.0)	12.0-14.0 (83-97)
2	130	130 (54.5)	165 (74.0)	(88.0)	88.0)	230 (110.0)	300 (149.0)	9.0-11.0 (62-76)
9	130	130 (54.5)	195 (90.5)		240 (115.5)	265 (129.5)	300 (149.0)	9.0-11.0 (62-76)
7	06	90 (32.0)	135 (57.0)	(88.0)	88.0)	265 (129.5)	360 (182.0)	12.0-14.0 (83-97)
<b>&amp;</b>	130	130 (54.5)	165 (74.0)	(88.0)	88.0)	265 (129.5)	360 (182.0)	9.0-11.0 (62-76)
6	06	90 (32.0)	155 (68.5)		240 (115.5)	265 (129.5)	300 (149.0)	12.0-14.0 (83-97)

Additional

Minimum Octane Number = 91 RON, 83 MON, 87  $\frac{R+M}{2}$ 

Benzene Content = 5% maximum

Oxidation Inhibitor = 5 lbs./1009 barrels DuPont No. 31 (or equivalent)

Contaminant Concentrations (maximum) = Pb, 0.05 g/gal

S, 0.04 wt %

P, 0.005 g/gal

\* ± 5°F (± 3°C)

\*\* ± 10°F (± 5.5°C)

TABLE 8-2

Test Design - Low Temperature Phase

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\* OL = Open Loop CL = Closed Loop

TABLE B-

Test Design - Intermediate Temperature Phase

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\* OL = Open Loop CL = Closed Loop

TABLE B-4
Estimation of Program Duration and Manpower Requirement

# Program Duration

	Days Required Per Test Phase
Preparation and Driver Selection	2
Testing	22
Weekend and Weather Allowance	_8
Total Days Per Phase	$31 = 4 \frac{1}{2}$ weeks

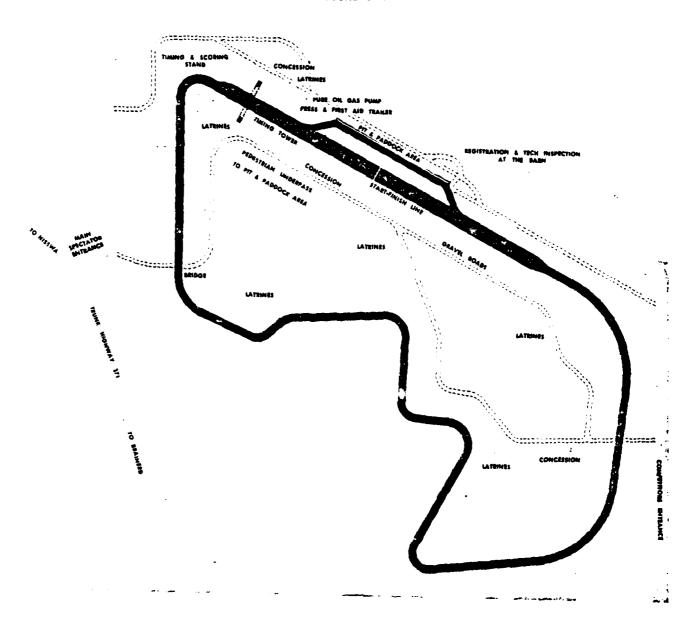
Total for Entire Program =  $2 \times 4 \cdot 1/2$  weeks = 9 weeks

# Manpower Requirement

	Number <u>Required</u>
Raters	2
Observers	2
Fueling and Supervision	1
Data Handling	<u>1</u>
Total	6

Total Manpower Requirement = 6 x 9 = 54 man-weeks

FIGURE B-1



Layout of the 3-Mile Donnybrooke Track

# CRC COLD START AND NARMUP DRIVEABILITY PROCEDURE

# TEST PROCEDURE AND DATA RECORDING

- A. Record all necessary test information at the top of the data sheet.
- B. Start engine per Owner's Manual Procedure. Record start time.
- C. If engine fails to start after 15 seconds of cranking, stop cranking and depress accelerator pedal to the floor once and release. Begin cranking and record total cranking time until engine starts.
- D. Record idle quality in "Neutral" or "Park" immediately after start; foot should be removed from accelerator pedal.
- E. If engine stalls, repeat Steps B and C. Record number of stalls and starting time of required restarts.
- F. Allow engine to idle 15 seconds. Apply brakes (right foot), shift to normal drive range, and record idle quality. If engine stalls, restart immediately. Do not record restart time. Record number of stalls. Idle 5 seconds in "Drive."

This completes the start-up portion of the procedure. Note that space on the data sheet has only been provided for two restart times at any idle condition. If three stalls occur at any condition, record the three stalls, restart (without recording time) and proceed to the next scheduled condition.

G. After 5 seconds in "Drive" (Step F), make a light throttle (Lt. th) acceleration from 0-25 mph at constant throttle opening beginning at the predetermined manifold vacuum.\* Cruise at 25 mph. At the 0.2 mile marker open throttle to the detent position and accelerate from 25 to 35 mph at constant throttle in high gear. Decelerate to a stop, and at the 0.3 mile marker make a WOT acceleration from 0 to 35 mph. Decelerate to 10 mph and at mile marker 0.4 accelerate at light throttle from 10 to 25 mph. Definitions of light throttle, detent, and WOT accelerations are attached.

<sup>\*</sup> Marked on vacuum gauge.

- H. During the above maneuvers observe and record the severity of any of the following malfunctions (see attached definitions):
  - 1. Hesitation
  - 2. Stumble
  - 3. Surge
  - 4. Stall
  - 5. Backfire

Record maneuvering stalls on the data sheet in the appropriate column: accelerating or decelerating. In addition, measure and record the time required to accelerate from 0-25 on the 0-25 mph maneuver.

- I. At the 0.5 mile marker, brake moderately to a stop on the right side of the roadway. Idle for 30 seconds in drive. Record idle quality and number of stalls.
- J. Perform Steps G, H, and I three times (1.5 miles). The mile marker for the beginning of each maneuver is indicated on the data sheet.
- K. At mile marker 1.5, after completing the 30 second idle, make a crowd acceleration (constant predetermined vacuum) from 0-45 mph. Four-tenths of a mile is provided for this maneuver. Decelerate from 45 to 25 mph at the 1.9 mile marker, and open throttle to detent position and accelerate from 25 to 35 mph. At 2.0 miles decelerate to a stop and accelerate from 10 to 25 mph at light throttle. Rate and record malfunctions in these maneuvers as in Step H. Measure and record the time required to travel the first 0.3 miles of the 0-45 mph crowd maneuver. Idle 30 seconds in drive as in Step I.
- L. Perform Step K three times. Appropriate mile markers for the start of each maneuver are shown on the data sheet.

#### DEFINITIONS AND EXPLANATIONS

#### Test Run

Operation of a car throughout the prescribed sequence of operating conditions and/or maneuvers for a single test fuel.

#### Maneuver

A specified single vehicle operation or change of operating conditions (such as idle, acceleration or cruise) that constitutes one segment of the driveability driving schedule.

## Cruise

Operation at a prescribed constant vehicle speed with a fixed throttle position on a level road.

## Wide Open Throttle (WOT) Acceleration

"Floorboard" acceleration through the gears from prescribed starting speed. Rate at which throttle is depressed is to be as fast as possible without producing tire squeal or appreciable slippage.

# Part Throttle (PT) Acceleration

An acceleration made at any defined throttle position, or consistent change in throttle position, less than WOT. Several PT accelerations are used. They are:

- Light Throttle (Lt th) All light throttle accelerations are begun by opening the throttle to an initial manifold vacuum and maintaining constant throttle position throughout the remainder of the acceleration. The vacuum selected is one in. Hg greater than the initial power cut-in vacuum obtained from carburetor flow curves. However, if a 0-25 mph light throttle maneuver (car warmed up) cannot be completed in 0.1 mile, vacuum is decreased in steps of one in. Hg until the 0-25 maneuver can be completed in 0.1 mile. The selected vacuum is posted in each car.
- 2. Crowd An acceleration made at a constant intake manifold vacuum. To maintain constant vacuum, the throttle opening must be continually increased with increasing engine speed. Crowd accelerations are performed at the same vacuum prescribed for the light throttle acceleration.
- 3. Detent All detent accelerations are begun by opening the throttle to the downshift position as indicated by transmission shift characteristic curves. Manifold vacuum corresponding to this point at 25 mph is posted in each car. Constant throttle position is maintained to 35 mph in this maneuver.

# Malfunctions

# 1. Stall

Any occasion during a test when the engine stops with the ignition on. Three types of stall, indicated by location on the data sheet, are:

- a. Stall; idle Any stall experienced when the vehicle is not in motion, or when a maneuver is not being attempted.
- b. Stall; maneuvering Any stall which occurs during a prescribed maneuver or attempt to maneuver.
- c. Stall; decelerating Any stall which occurs while decelerating between maneuvers.

# 2. Idle Roughness

An evaluation of the idle quality or degree of smoothness while the engine is idling.

#### 3. Backfire

An explosion in the induction or exhaust system.

#### 4. Hesitation

A temporary lack of vehicle response to opening of the throttle.

#### 5. Stumble

A short, sharp reduction in acceleration after the vehicle is in motion.

#### 6. Surge

Cyclic power fluctuations occurring during acceleration or cruise.

# Malfunction Severity Ratings

The number of stalls encountered during any maneuver are to be listed in the appropriate data sheet column. Each of the other malfunctions must be rated by severity and the letter designation entered on the data sheet. The following definitions of severity are to be applied in making such ratings:

- Trace (T) A level of malfunction severity that is just discernible to a test driver but not to most laymen.
- 2. Moderate (M) A level of malfunction severity that is probably noticeable to the average layman.
- 3. Heavy (H) A level of malfunction severity that is pronounced and obvious to both test driver and layman.

Enter a T, M or H in the appropriate data block to indicate both the occurrence of the malfunction and its severity. More than one type of malfunction may be recorded on each line. If no malfunctions occur, enter a dash (-) to indicate that the maneuver was performed and operation was satisfactory during that maneuver.

# CRC driveability data sheet

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# DEMERIT CALCULATION SYSTEM

A numerical value for driveability during the CRC test is obtained by assigning demerits to operating malfunctions as shown in Table B-5. repending on the type of malfunction, demerits are assigned in various ways. Demerits for poor starting are obtained by subtracting two seconds from the measured starting time. The number of stalls which occur during idle as well as during driving maneuvers are counted separately and assigned demerits as shown in Table B-5. The multiplying factors of 8 and 32 for idle and maneuvering stalls, respectively, account for the fact that stalls are very undesirable, expecially during car maneuvers. Other malfunctions, such as hesitation, stumble, surge, idle roughness, and backfire, are rated subjectively by the driver on a scale of trace, moderate, or heavy. For these malfunctions, a certain number of demerits is assigned to each of the subjective ratings. However, since all malfunctions are not of equal importance, the demerits are multiplied by the weighting factors shown in Table B-5 to yield weighted demerits. Finally, weighted demerits, demerits for stalls, and demerits for poor starting are summed to obtained total weighted demerits (TWD), which are used as an indication of driveability during the test. As driveability deteriorates, TWD increase.

A restriction has been applied in the totaling of demerits to insure that a stall results in the highest possible number of demerits within a given maneuver. When more than one malfunction occurs during a maneuver, demerits are counted for only the malfunction which had the largest number of weighted demerits. Another restriction was that for each idle period, no more than 3 idle stalls were counted.

# TABLE B-5

# Method for Calculating Total Weighted Demerits (TWD)

Demerits for Poor Starting:

Demerits = starting time(s) - 2

Demerits for Stalls:

Demerits = (no. of idle stalls)  $\times$  8 + (no. of maneuvering stalls)  $\times$  32

Demerits for Malfunctions Rated Subjectively:

Demerits for Subjective Ratings

Trace = 1

Moderate = 2

Heavy = 4

Weighting Factors for Each Malfunction

Idle Roughness

Surge

Backfire, Stumble, Hesitation = 6

Weighted Demerits = Demerits x Weighting Factor

#### Calculation:

Total Weighted Demerits = Weighted Demerits + Demerits for Stalls +

Demerits for Poor Starting

Note: When more than one malfunction occurs in a driving maneuver, only the malfunction giving the highest weighted demerits is counted.

# ON-SITE CAR PREPARATION SCHEDULE

- a. Install vacuum gauges in cars.
- b. Drive cars to determine light-throttle and detent vacuums to be used for tests.

Light-throttle vacuum: This vacuum will be determined by performing accelerations from 0 to 25 mph. Warm up the car first. Then select a vacuum, start the acceleration from 0 at this vacuum, and maintain constant throttle position. Repeat until a vacuum is found which allows a 0 to 25 mph acceleration to be made in 0.05 to 0.1 miles. Mark this vacuum on vacuum gauge.

Detent vacuum: Cruise at 25 mph. Open the throttle to a preselected vacuum and determine if a downshift has occurred. Repeat to find the minimum vacuum level which can be achieved without a downshift. The vacuum for the detent acceleration should be 1 in. Hg above this minimum vacuum. Mark on vacuum gauge.

c. Install identifying numbers on cars and fuel cans.

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APPENDIX C

DATA LISTING

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AD-A119 146 COORDINATING RESEARCH COUNCIL INC ATLANTA BA

EFFECTS OF FUEL VOLATILITY ON DRIVEABILITY OF 1980 MODEL CARS A--ETC(U)

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TABLE C-2. RAW TOTAL WEIGHTED DEMERITS SUMMARY

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# APPENDIX D

DEVELOPMENT OF RATER CORRECTION FACTORS
AND PRECISION STATEMENTS

#### DEVELOPMENT OF RATER CORRECTION FACTORS AND PRECISION STATEMENTS

STEP 1: Determine offsets between raters on-site simultaneously.

To determine these individual rater offsets, the data for each phase were broken into four segments based on test cars and test days. In both phases, cars 1-8 were grouped and cars 9-16 were grouped, because all cars in each group followed the same fueling schedule. The data for the low temperature phase were also divided by grouping days 1-11 and days 15-26, because Raters A and B were on-site in the first portion, and Raters C and D were on-site in the latter days. Days 12, 13, and 14 were eliminated to avoid trying to identify day-by-fuel interactions on these days which would not occur on other days. The data for the intermediate temperature phase were divided at day 14, when the rater teams changed. The following eight linear regressions were run to determine the individual rater offsets:

DATA

EQUATION FORM\*

EQUATION NUMBER

## LOW TEMPERATURE PHASE

Days 1-11

Cars 1-8

Raw TWD = 
$$b_0 + b_1$$
 (Car 2) +  $b_2$  (Car 3) (1)

+  $b_3$  (Car 4) +  $b_4$  (Car 5)

+  $b_5$  (Car 6) +  $b_6$  (Car 7)

+  $b_7$  (Car 8) +  $b_8$  (Rater B)

+  $b_9$  (Day 2) +  $b_{10}$  (Day 3)

+  $b_{11}$  (Day 4) +  $b_{12}$  (Day 5)

+  $b_{13}$  (Day 6) +  $b_{14}$  (Day 7)

+  $b_{15}$  (Day 8) +  $b_{16}$  (Day 9)

+  $b_{17}$  (Day 10) +  $b_{18}$  (Day 11)

Base Case is Car 1, Rater A, Day 1

<sup>\*(</sup>Car "n"), (Rater "m"), and (Day "n") are dummy variables whose value is 1.0 for observations with car "n", Rater "m", and day "n", and whose value is 0.0 otherwise. For example, for car 2 (Rater B) on day 2, the equation becomes:  $TWD = b_0 + b_1 + b_8 + b_9$ .

DATA

## EQUATION FORM

EQUATION NUMBER

# LOW TEMPERATURE PHASE (Continued)

Days 1-11 (Continued)

Cars 9-16 Raw TWD = 
$$b_0 + b_1$$
 (Car 10) (2)  
+ ...  $b_7$  (Car 16)  
+  $b_8$  (Rater B) +  $b_9$  (Day 2)  
+ ...  $b_{18}$  (Day 11) ...

Base Case is Car 9, Rater A, Day 1

Days 15-26

Cars 1-8 Raw TWD = 
$$b_0 + b_1$$
 (Car 2) (3)  
+ ...  $b_7$  (Car 8)  
+  $b_8$  (Rater C) +  $b_9$  (Day 16)  
+ ...  $b_{19}$  (Day 26)

Base Case is Car 1, Rater D, Day 15

Cars 9-16 Raw TWD = 
$$b_0 + b_1$$
 (Car 10) (4)  
+ ...  $b_7$  (Car 16)  
+  $b_8$  (Rater C) +  $b_9$  (Day 16)  
+ ...  $b_{19}$  (Day 26)

Base Case is Car 9, Rater D, Day 15

EQUATION NUMBER

# INTERMEDIATE TEMPERATURE PHASE

# Days 1-14

Cars 1-8 Raw TWD = 
$$b_0 + b_1$$
 (Car 2) (5)  
+ ...  $b_7$  (Car 8)  
+  $b_8$  (Rater E)  
+  $b_9$  (Rater G) +  $b_{10}$  (Day 2)  
+ ...  $b_{22}$  (Day 14)

Base Case is Car 1, Rater D, Day 1

Cars 9-16 Raw TWD = 
$$b_0 + b_1$$
 (Car 10) (6)  
+ ...  $b_7$  (Car 16)  
+  $b_8$  (Rater E)  
+  $b_9$  (Rater G) +  $b_{10}$  (Day 2)  
+ ...  $b_{22}$  (Day 14)

Base Case is Car 9, Rater D, Day 1

# Days 15-26

Cars 1-8 Raw TWD = 
$$b_0 + b_1$$
 (Car 2) (7)  
+ ...  $b_7$  (Car 8)  
+  $b_8$  (Rater I) +  $b_9$  (Day 16)  
+ ...  $b_{19}$  (Day 26)

Base Case is Car 1, Rater H, Day 15

# INTERMEDIATE TEMPERATURE PHASE (Continued)

Days 15-26 (Continued)

Cars 9-16 Raw TWD = 
$$b_0 + b_1$$
 (Car 10) (8)  
+ ...  $b_7$  (Car 16)  
+  $b_8$  (Rater I) +  $b_9$  (Day 16)  
+ ...  $b_{19}$  (Day 26)

Base Case is Car 9, Rater H, Day 15

After a few iterations, it was discovered that three observations were outliers. They were observations 255, 878, and 901. Detailed results of these regressions after eliminating the outliers are shown in Table D-1. Pertinent results for rater corrections are:

Base Rater	Variable	Coeff.*	s.e. of Coeff.	t-Value**	Coeff.*	s.e. of Coeff.	t-Value**
		E	quation 1	····	E	quation 2	
Rater A	b <sub>8</sub> (Rater B)	41.7	25.3	1.6	21.7	30.6	0.7
		E	quation 3		E	quation 4	
Rater D	b <sub>8</sub> (Rater C)	33.7	8.4	4.0	14.9	16.9	0.9
		E	quation 5		E	quation 6	
Rater D	b <sub>8</sub> (Rater E)	11.1	16.0	0.7	27.8	17.5	1.6
Rater D	b <sub>8</sub> (Rater G)	-15.0	12.6	1.2	-21.1	15.2	1.4
		E		E	quation 8		
Rater H	b <sub>8</sub> (Rater I)	35.3	12.3	2.9	36.1	15.8	2.3

<sup>\*</sup>Coefficient is the  $\Delta$  TWD between Base Rater and Rater shown in the "Variable" column. \*\*Must be  $\geq$  2.0 to assure coefficient is not zero at the 95% confidence level.

From Equations 1 and 2:

Average Rater A TWD - Average Rater B TWD<sup>(1)</sup> = -32.7  
Standard error of 
$$\triangle$$
 TWD<sup>(2)</sup> = 19.2  
t-Value = 1.7

From Equations 3 and 4:

Average Rater D TWD - Average Rater C TWD<sup>(1)</sup> = -30.0  
Standard error of 
$$\triangle$$
 TWD<sup>(2)</sup> = 7.5  
t-Value = 4.0

From Equations 5 and 6:

Average Rater D TWD - Average Rater E TWD<sup>(1)</sup> = -18.7  
Standard error of 
$$\triangle$$
 TWD<sup>(2)</sup> = 11.8  
t-Value = 1.6

Average Rater D TWD - Average Rater G TWD<sup>(1)</sup> = 17.5  
Standard error of 
$$\triangle$$
 TWD<sup>(2)</sup> = 9.7  
t-Value = 1.8

From Equations 7 and 8:

Average Rater H TWD - Average Rater I TWD<sup>(1)</sup> = -35.6  
Standard error of 
$$\triangle$$
 TWD<sup>(2)</sup> = 9.7  
t-Value = 3.7

$$= \frac{\frac{\Delta \text{ TWD}_{1}}{(\text{s.e.}_{1})^{2}} + \frac{\Delta \text{ TWD}_{2}}{(\text{s.e.}_{2})^{2}}}{\frac{1}{(\text{s.e.}_{1})^{2}} + \frac{1}{(\text{s.e.}_{2})^{2}}}$$

(2) Standard error of wt. average TWD = 
$$\left[ \sqrt{\left(\frac{1}{s.e._1}\right)^2 + \left(\frac{1}{s.e._2}\right)^2} \right]^{-1}$$

# STEP 2: Determine offsets between raters not on-site simultaneously.

The second step in the rater correction process was to determine differences between raters on-site during the first and last half of each program phase. Recall that in regressions 1-8, offsets from a base case were calculated for each car, rater, and day. The day offsets are actually the day-to-day (or fuel-to-fuel) differences by the base rater averaged across all cars without variability due to cars or raters. To compare base raters (i.e., A and D in the low temperature phase, and D and H in the intermediate temperature phase), results of regressions 1-8 were used as shown in Table D-2 to calculate daily TWD. At this point, an investigation was included to determine whether ambient temperature was a significant variable within the program phases. The data for each phase were divided into two segments based again on test cars. One grouping was cars 1-8 and another grouping was cars 9-16. In the low temperature phase, data for test days 12, 13, and 14 were again eliminated. The four applicable equations are:

DATA

**EQUATION FORM** 

EQUATION NUMBER

#### LOW TEMPERATURE PHASE

Cars 1-8 Day Offset TWD = 
$$b_0 + b_1$$
 (Rater D) (9)  
+  $b_2$  (Soak Temperature)  
+  $b_3$  (Fuel 1) +  $b_4$  (Fuel 3)  
+ ...  $b_{10}$  (Fuel 9)

Base Case is Rater A and Fuel 2

Cars 9-16 Day Offset TWD = 
$$b_0 + b_1$$
 (Rater D) (10)  
+  $b_2$  (Soak Temperature)  
+  $b_3$  (Fuel 1) +  $b_4$  (Fuel 3)  
+ ...  $b_{10}$  (Fuel 9)

Base Case is Rater A and Fuel 2

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